

The Effects of Top of Rail Materials and Rail Grinding on Head Hardened Rail

Presented by

Nicholas Dryer BNSF Railway

Dr. Mark Richards EVRAZ North America

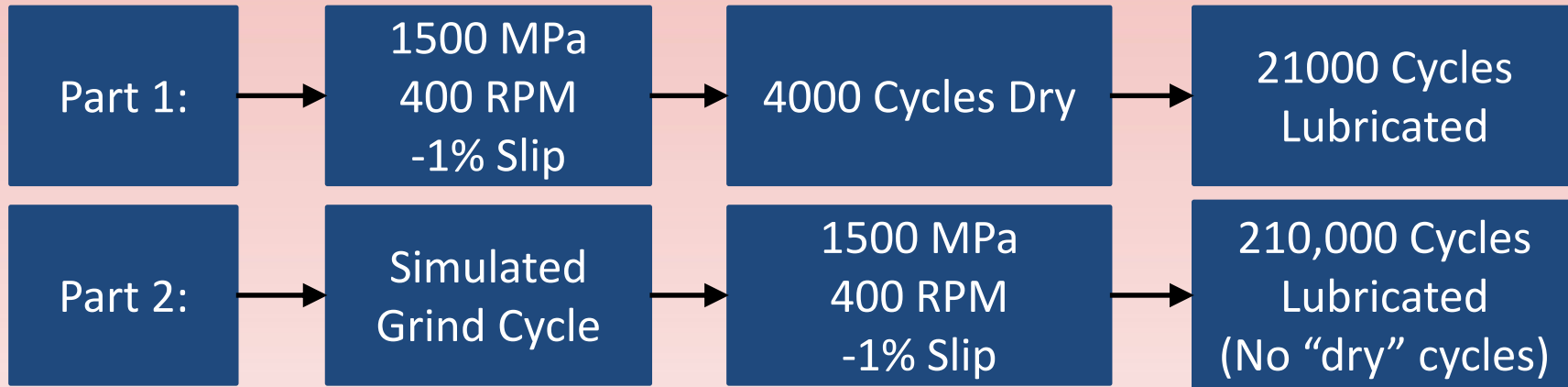


Background

- Previous research
 - Hardwick/Sheffield
 - Previous research indicated that using some products may have adverse effects on the rail, revenue service experience with these TOR materials did not typically manifest the previous results
- One more layer of real world approach
 - Class I railroads grind rail.
 - Would this mitigate crack propagation due to hydro-pressurization build up?



Test Outline



TOR Friction Modifier Materials

Water	Water Based TOR	Synthetic TOR
(1) - 1 Drop/second (0.53 ml/min)	0.05 ml/min	0.05 ml/min
(~0.497 g/min)	~0.046 g/min	0.038 g/min
(2) - Low Flow Rate (0.05 ml/min)	~0.055 g/ 500 cycles	~0.048 g/ 500 cycles
(~0.044 g/min)		



RCF Initialization

- Rolling Contact Fatigue

- Reproduce the conditions of Hardwick at University of Sheffield
- Sample geometry and twin disc machine parameters are modified to allow increased contact pressures and rotational speed
- Close control of slip ratio (creepage) and applied loads (contact pressure)
- Lubrication is applied after a dry cycle break in period

- 1500 MPa contact pressure
where:

$$P_{max} = \frac{2F}{\pi aL}$$

- F is the Force set point
- a is contact patch dimension
- L is the contact length

- -1% slip ratio (i.e. $V_{Wheel} > V_{Rail}$)
where:

$$\frac{Slip\ Ratio}{100\%} = \frac{2 \times (V_{Rail} - V_{Wheel})}{V_{Rail} + V_{Wheel}}$$

- V_{Rail} is the rail surface velocity
- V_{Wheel} is the wheel surface velocity



Test Materials

Rail Sample

Head Hardened

Rail: ~400 HB

$\varnothing = 45 \text{ mm}$

3.5 mm length



Wheel Surrogate

Head Hardened IH

Rail ~360 HB

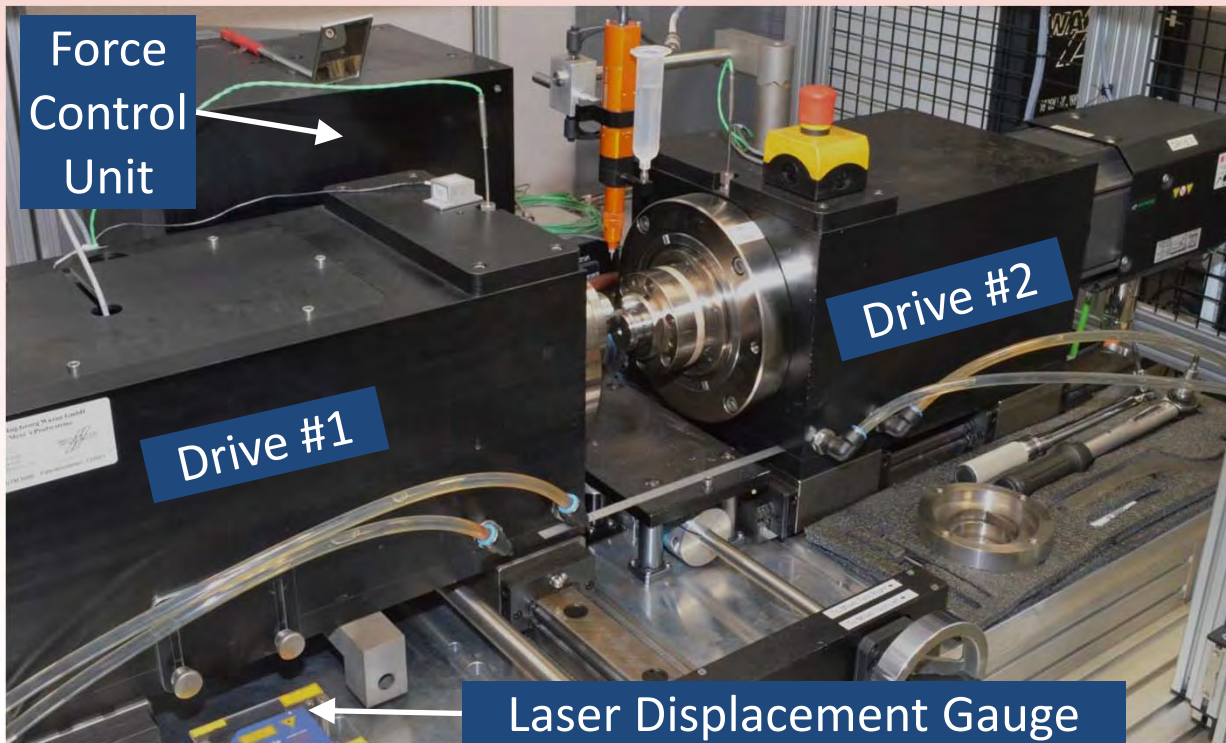
$\varnothing = 46.2 \text{ mm}$

20 mm length



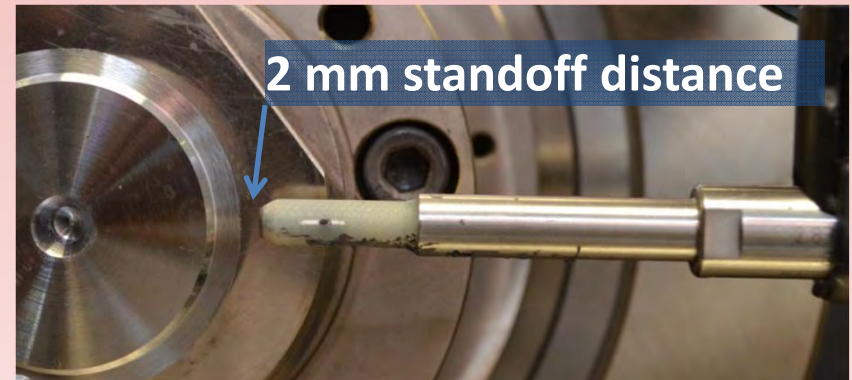
Wazau UTM 5000

- 5000 N Force Capacity
- 50 N-m Torque Capacity
- 3000 RPM, independent drive spindles
- *in-situ* eddy current
- Lubrication “drip” system - flow rate control



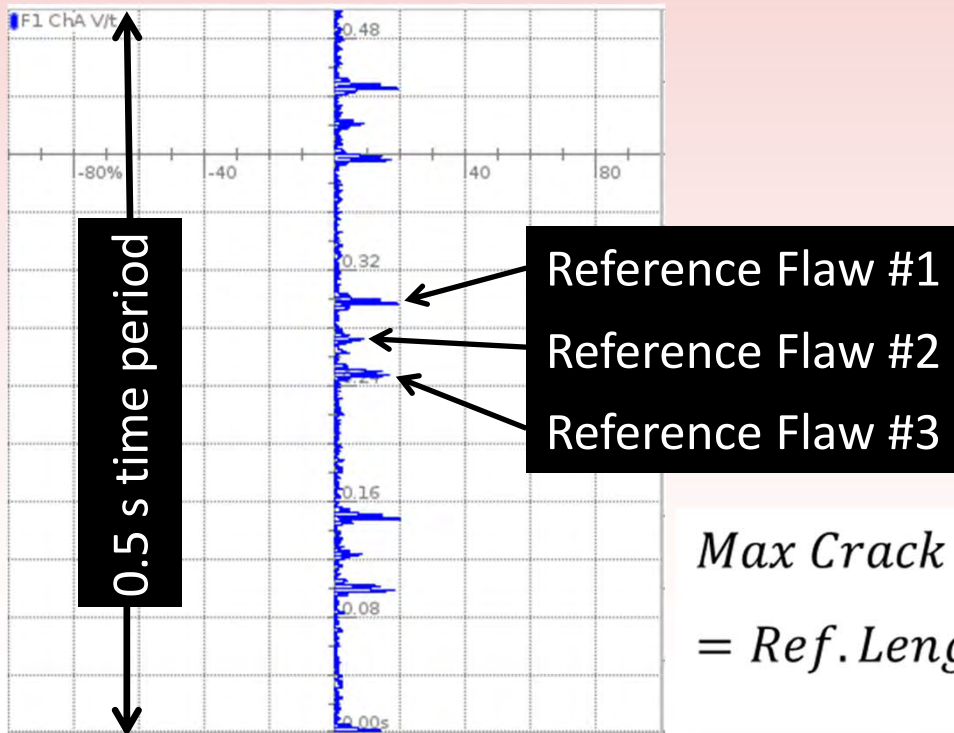
Eddy Current

- Constant Position for duration of test
- Calibrated against reference sample
 - 3 EDM reference flaws
 - 0.5 mm, perpendicular



Typical 3-4 repeated signals in 0.5 s

- Measure the largest peaks in:
 - Top/Middle/Bottom 3rds
- Used same feature if identifiable



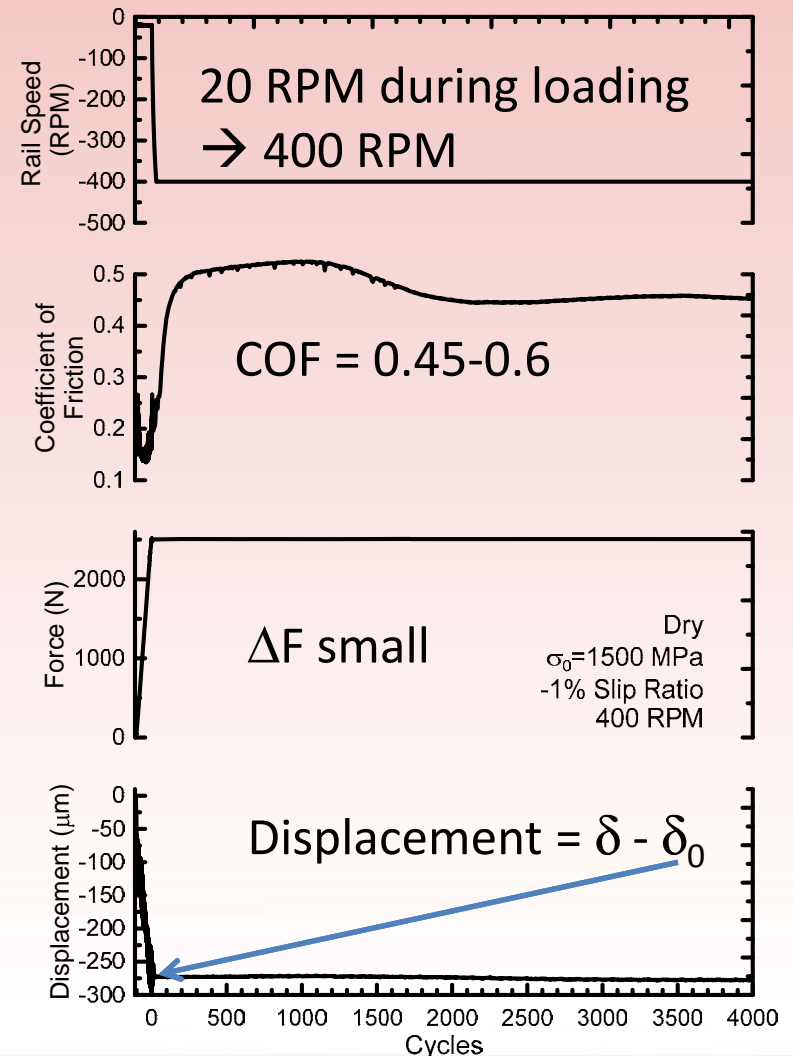
Max Crack Length

$$= \text{Ref. Length} \times \frac{\text{Ave. of 3 measurements}}{\text{Ave. of Calibration measurements}}$$



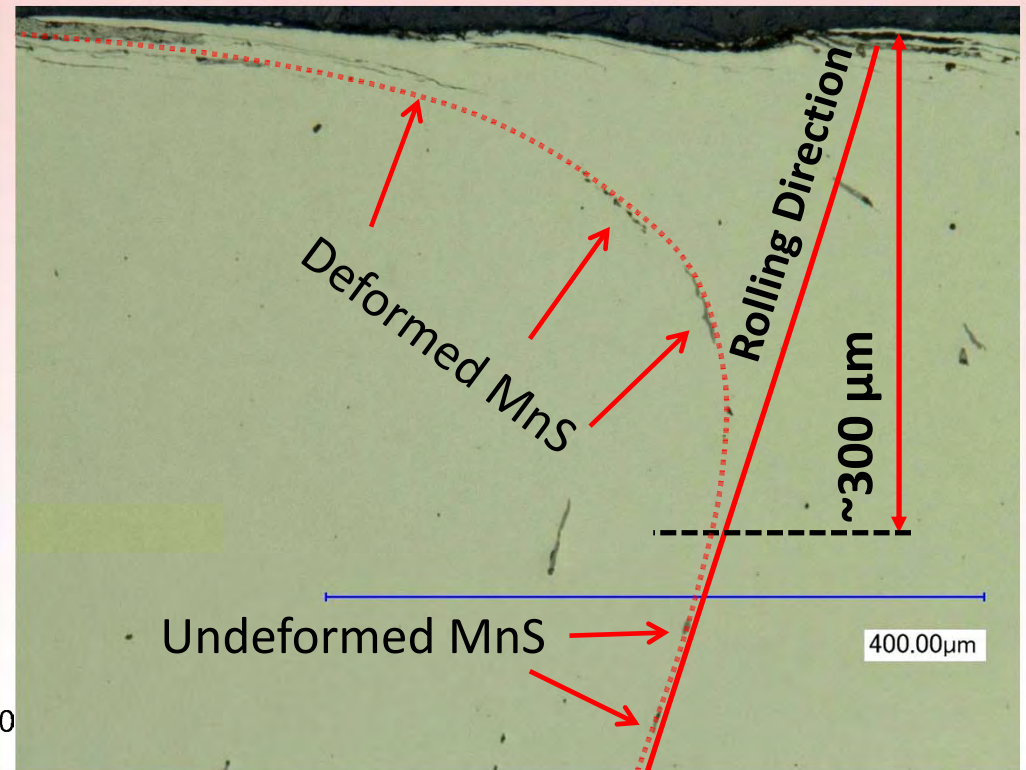
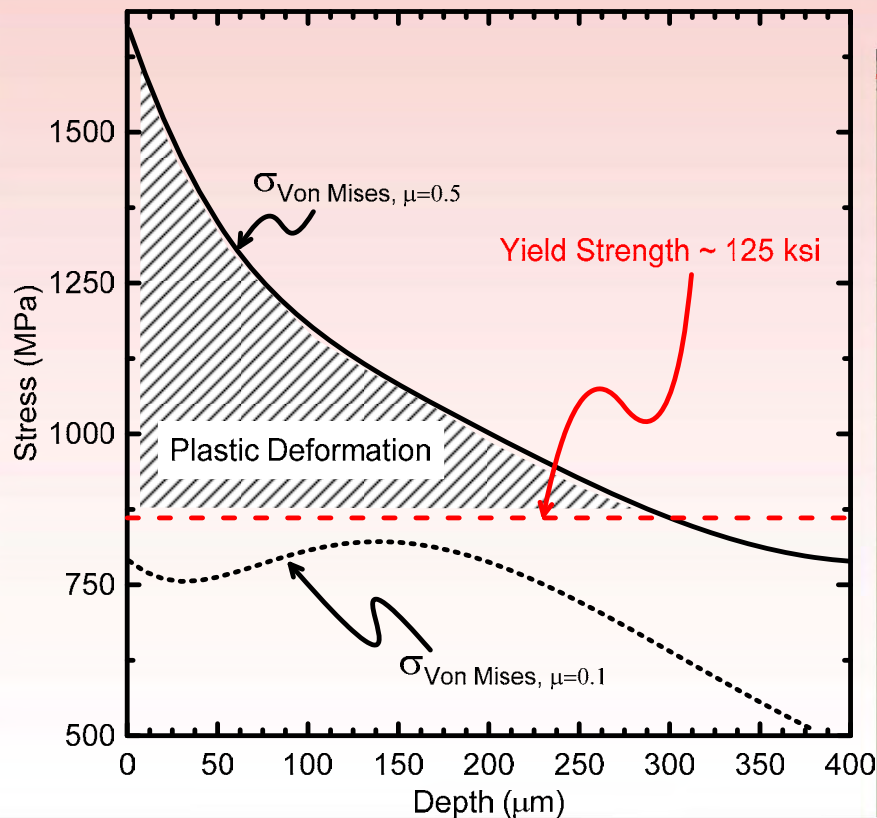
4000 Cycle, Unlubricated “Dry” Portion

- Rolling Contact Fatigue - Dry
 - Unlubricated portion to
 - Break-in the sample surface
 - Produce surface damage
 - High Coefficient of Friction (0.45-0.6)
 - Low speed during force ramping to limit “uncounted” dry cycles
 - Cycle count and displacement based upon achieving steady state force level



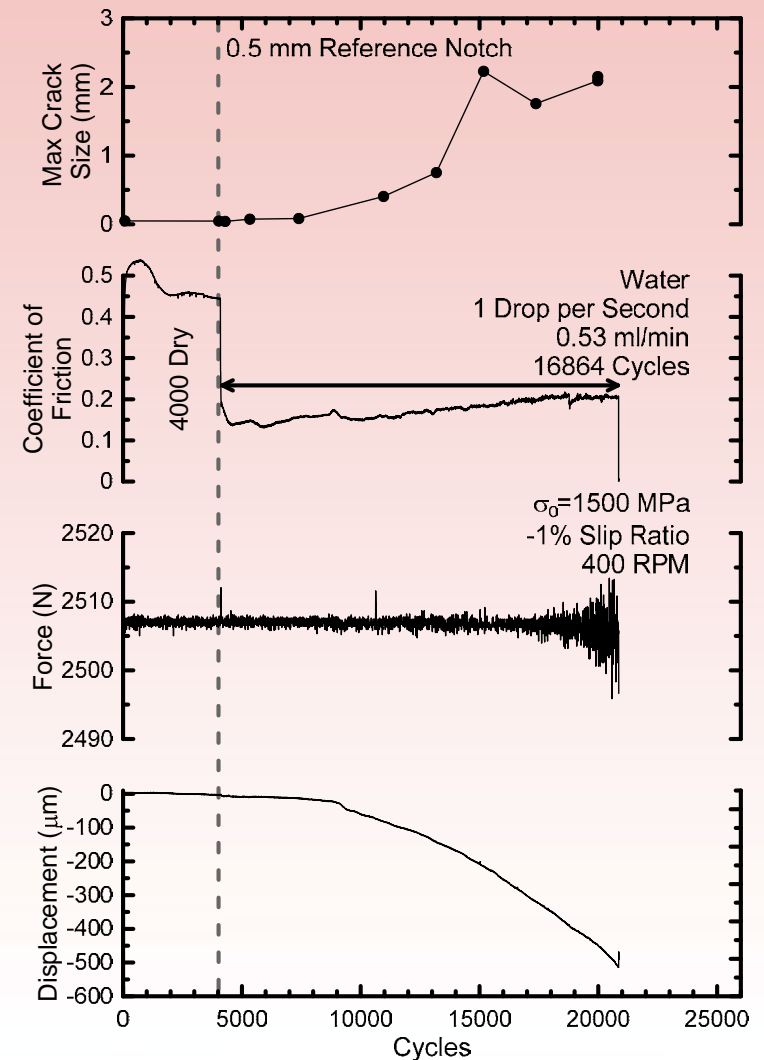
Contact Stress

- Friction introduces surface traction
- Shear component that greatly modifies the $\sigma_{\text{Von Mises}}$
- Dry cycles COF~0.45-0.6, Lubricated COF ~0.1
- Dry cycles cause severe surface damage / deformation

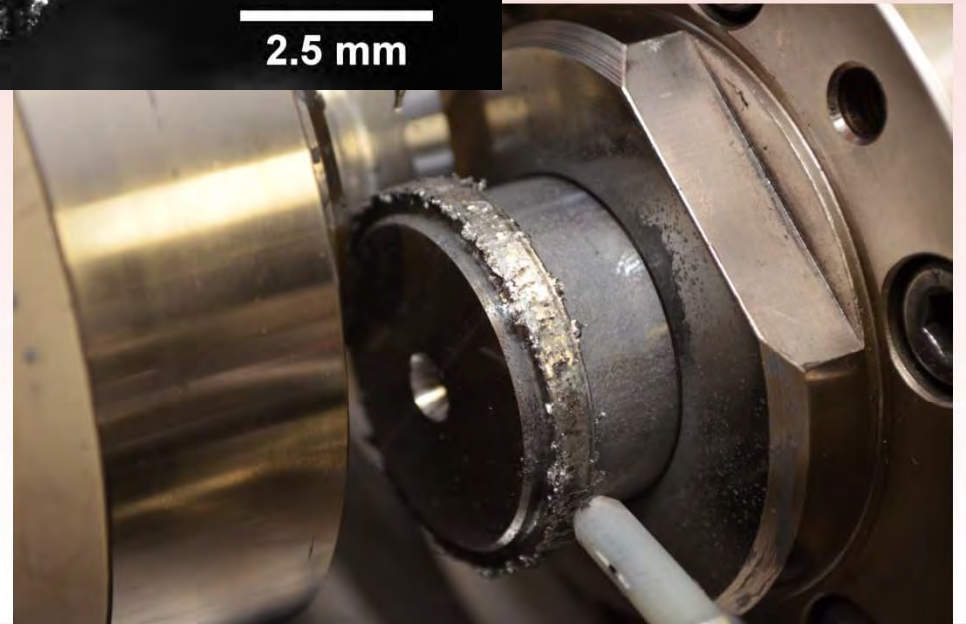
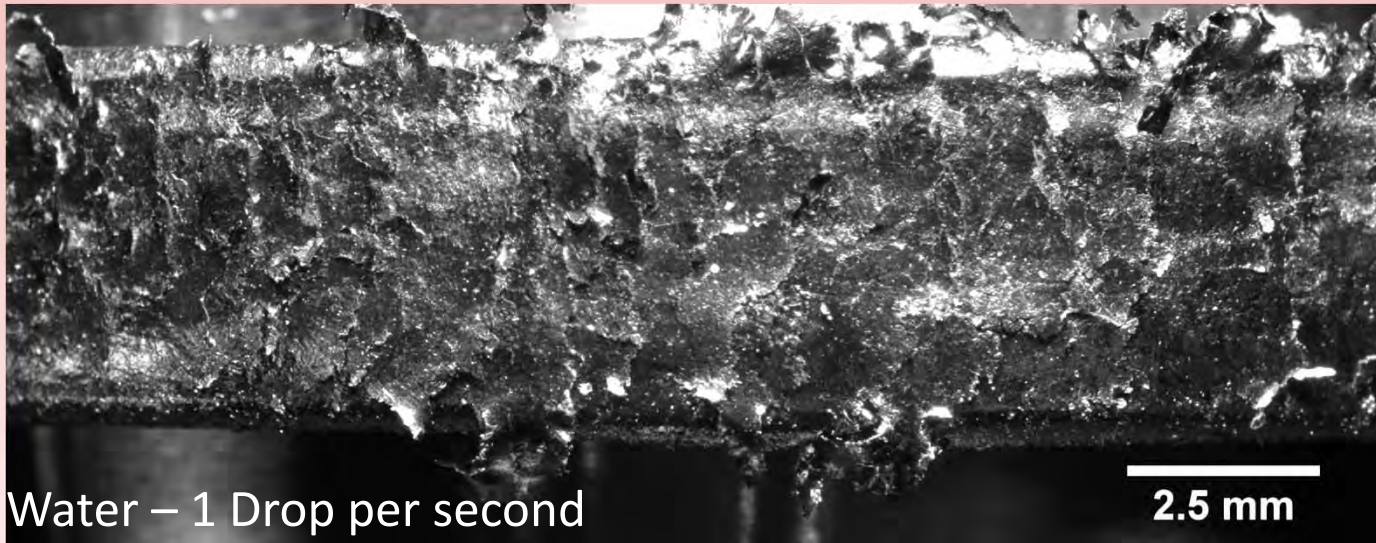


Water Part 1 – 1 Drop per second

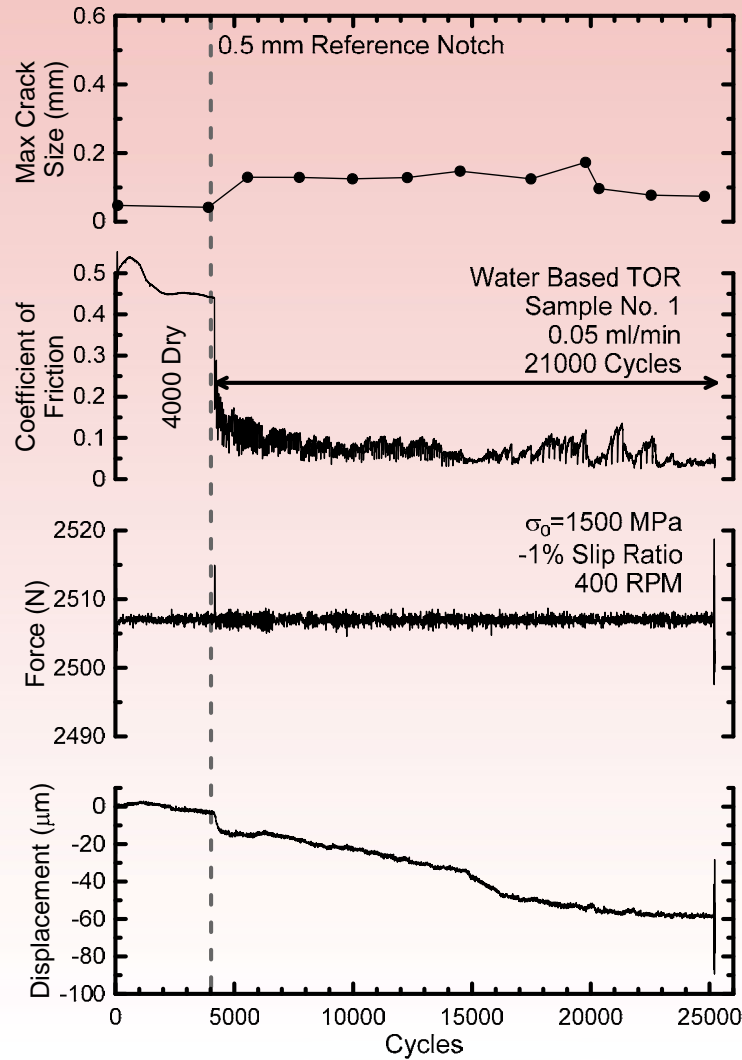
- RCF Data
 - Max Crack Size based upon Eddy Current Signal
 - Coefficient of Friction
 - Dry (0.45-0.6) → Lubricated (0.1)
 - Force is constant
 - Displacement → estimate of material loss



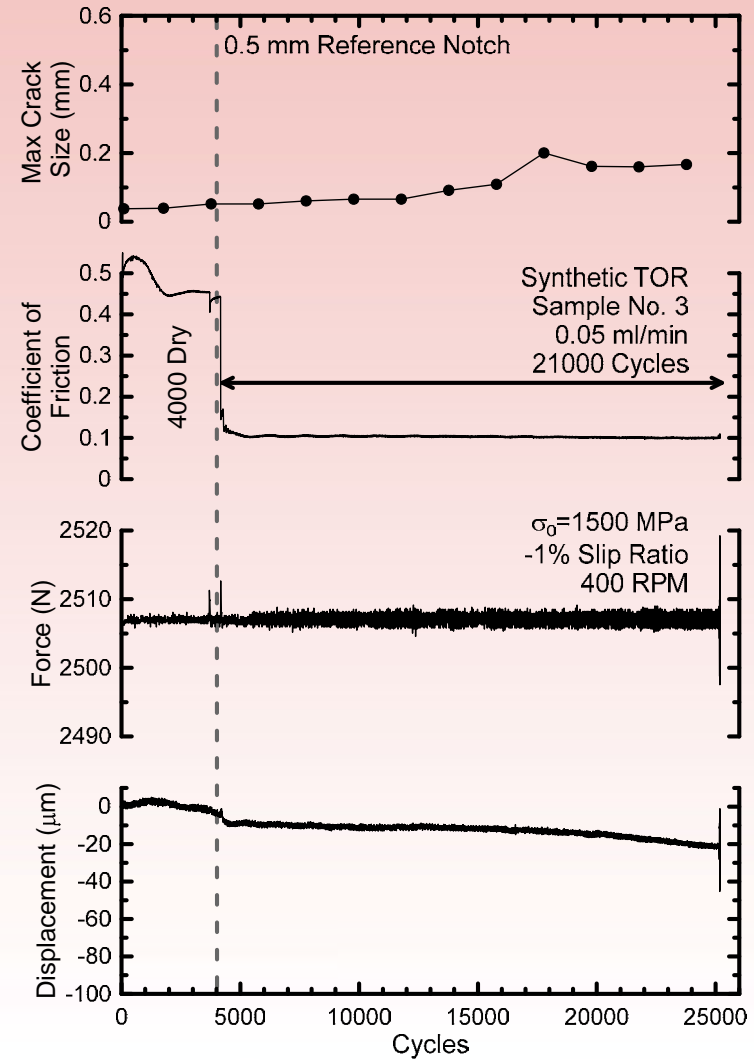
Water - 1 Drop per second



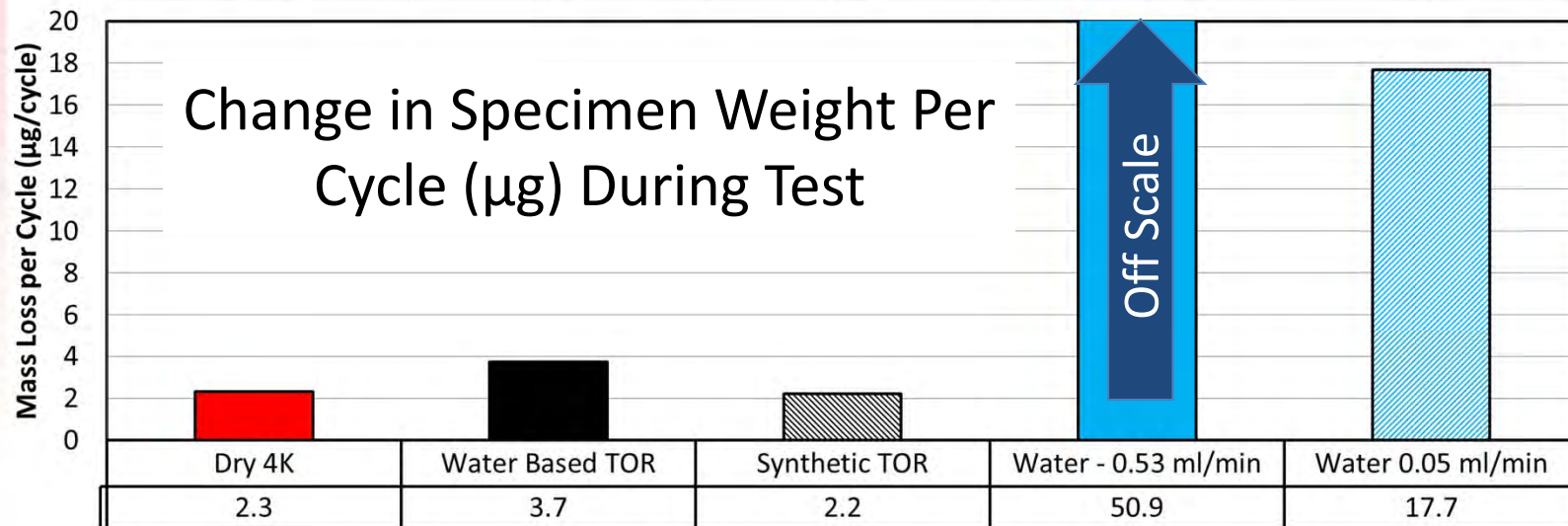
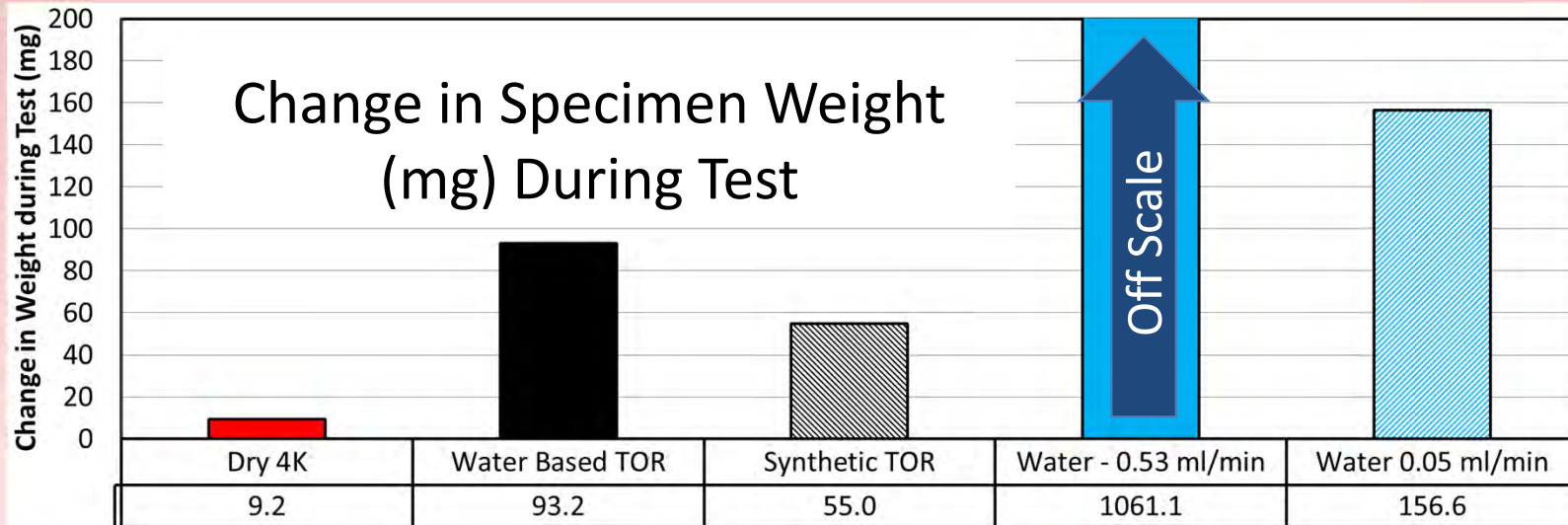
Water Based TOR



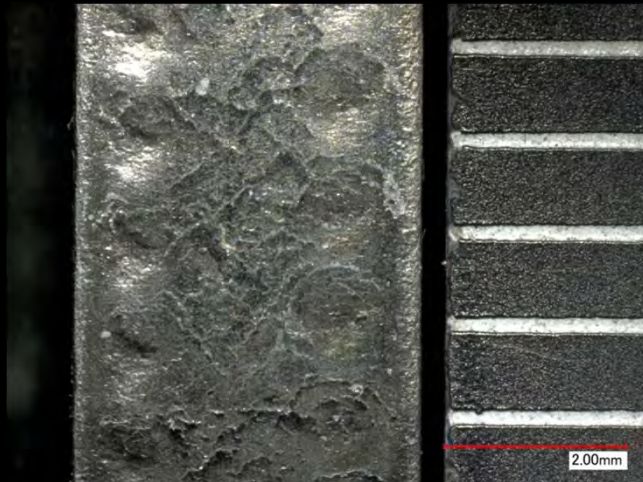
Synthetic TOR



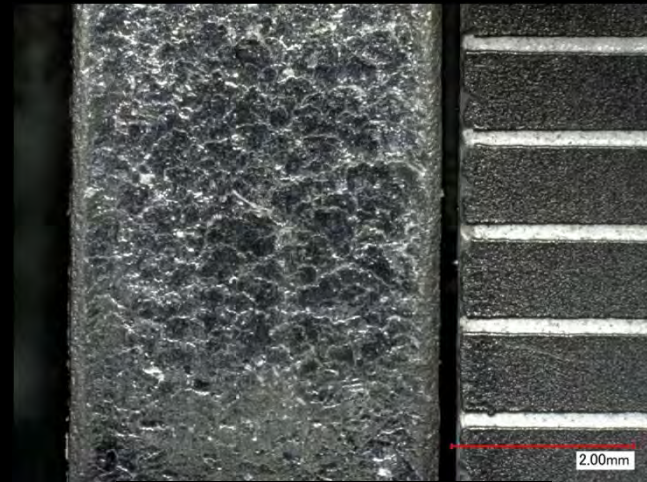
Material Loss – Part 1



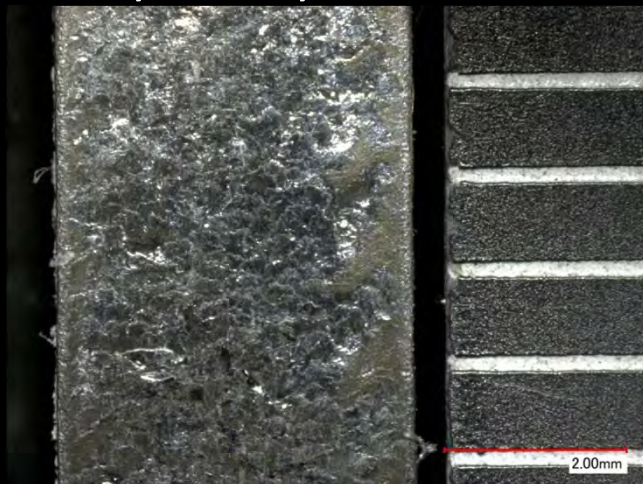
Part 1 – 4K Dry, 21K Lubricated



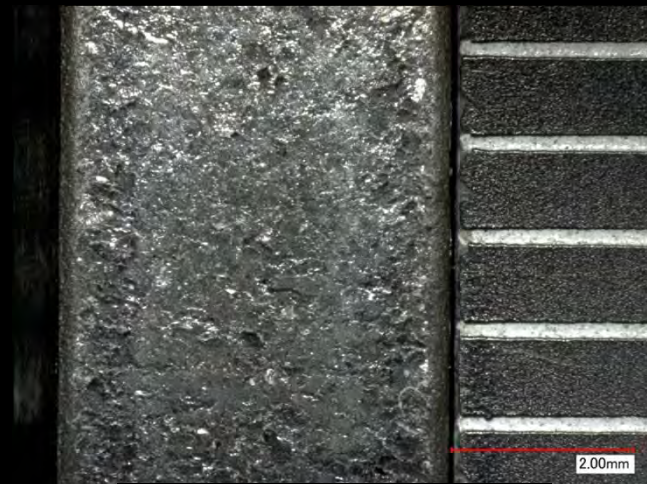
Dry – 4K cycles



Water Based TOR

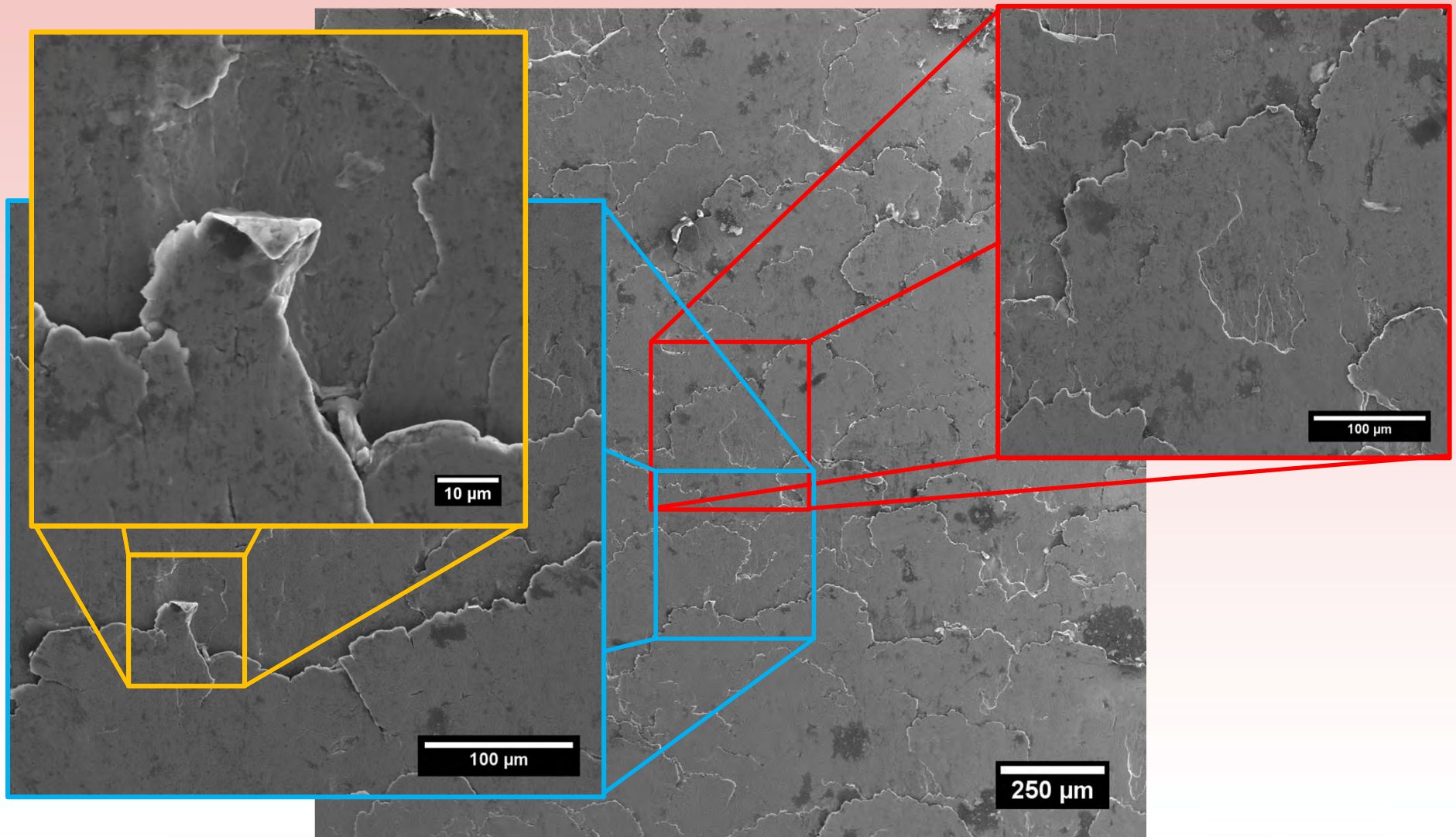


Synthetic TOR

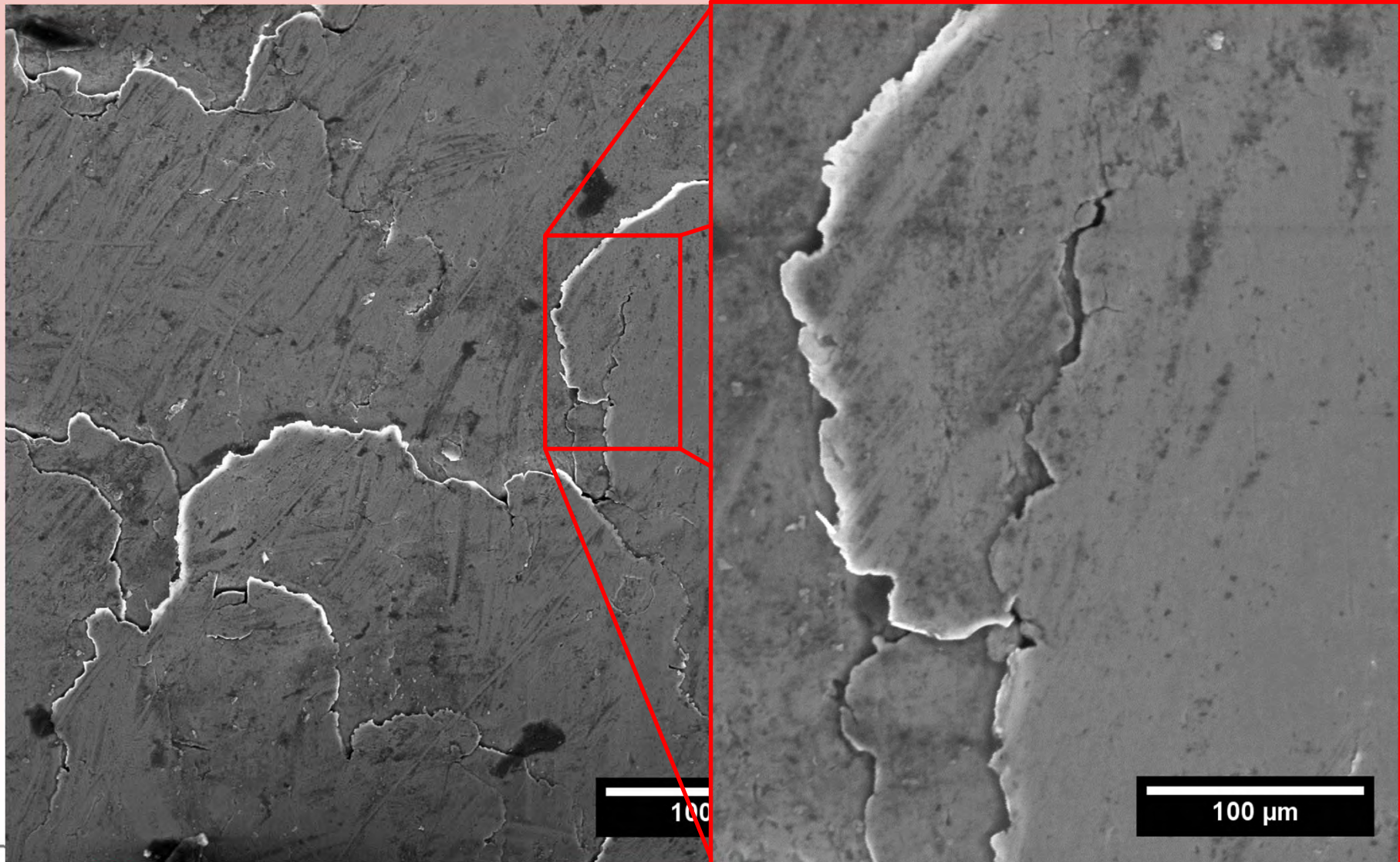


Water 0.05 ml/min

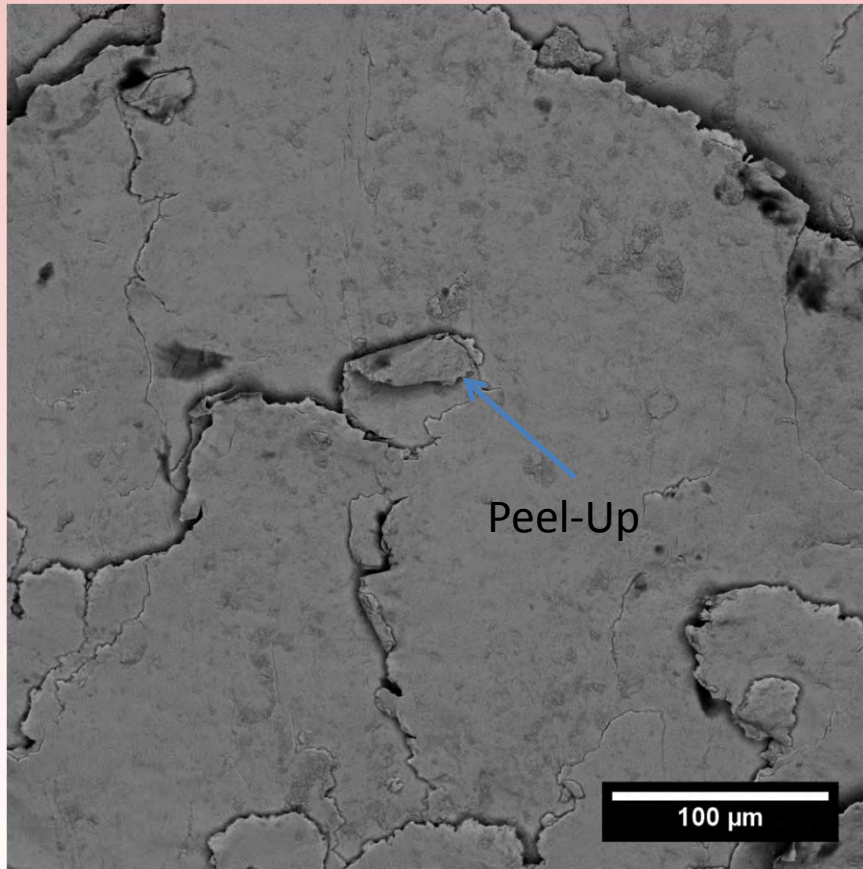
Water Based TOR



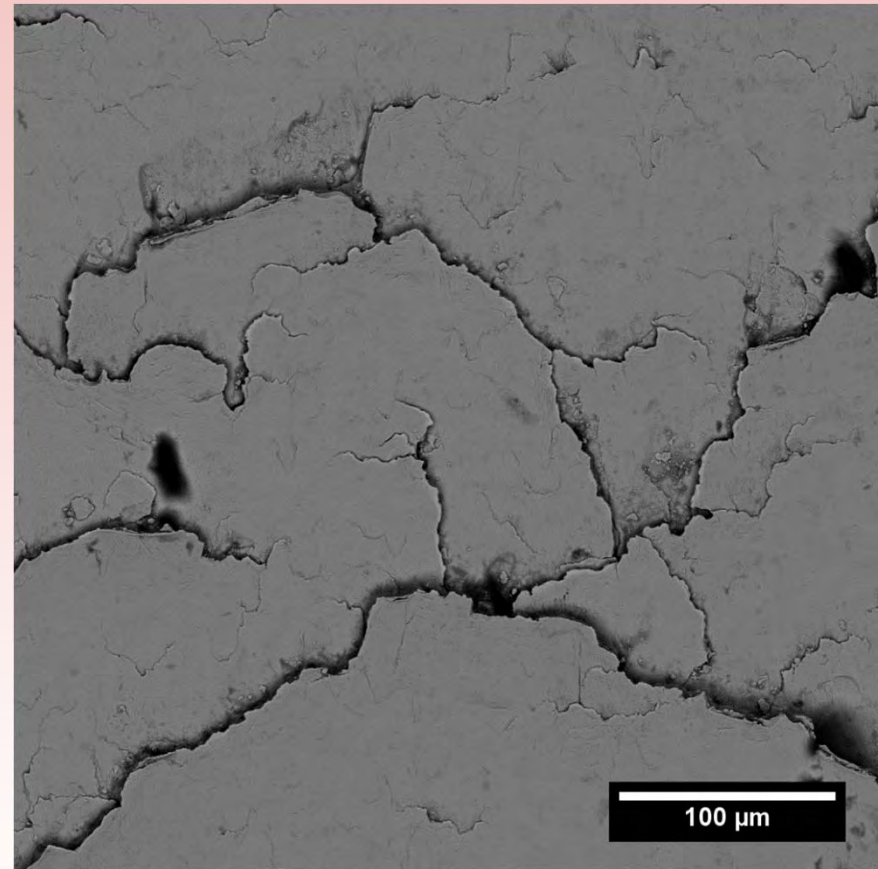
Synthetic TOR



Water and Dry 4k



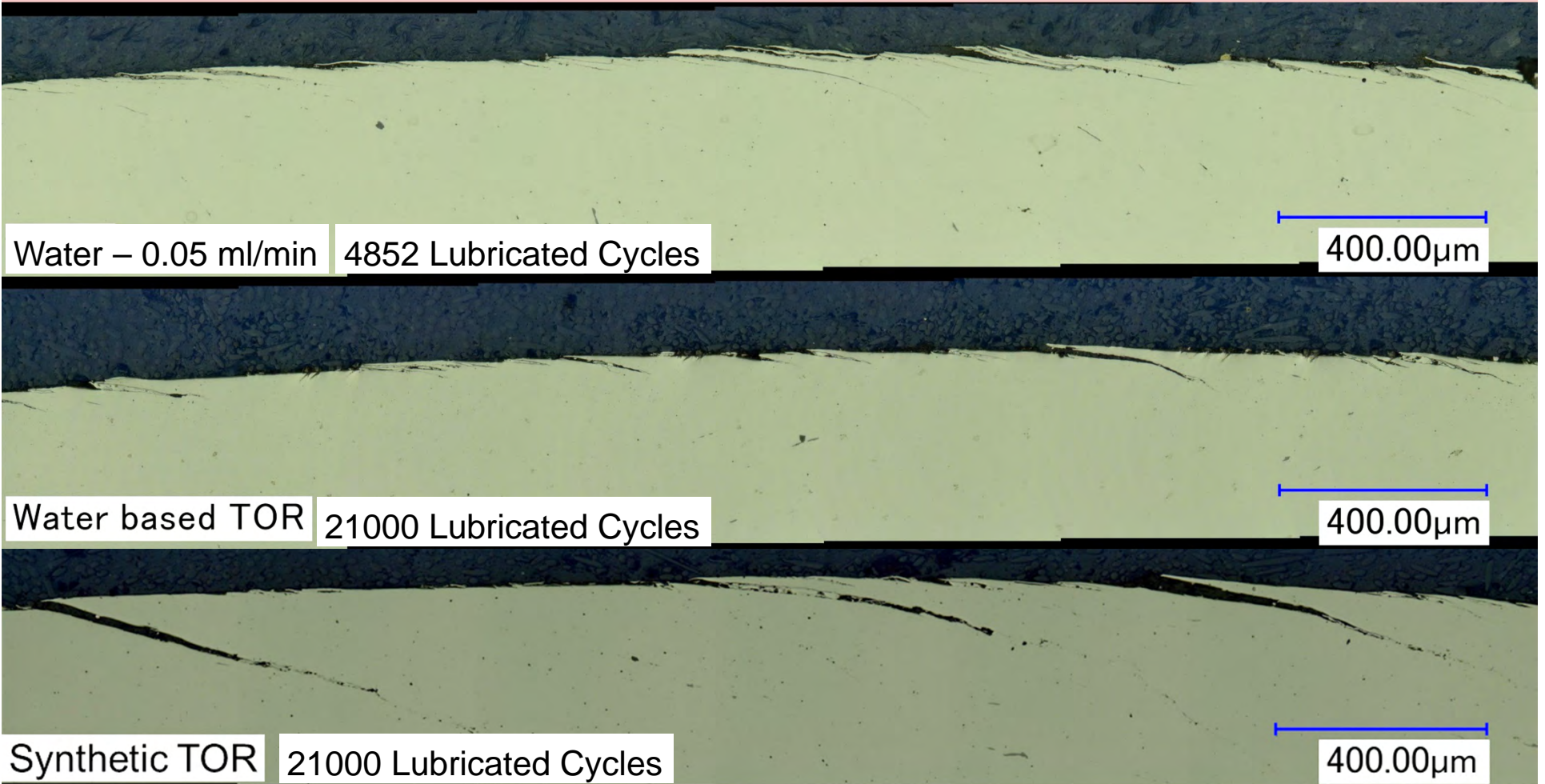
Water – 0.05 ml/min



Dry – 4K



Metallographic RCF Crack Comparison



RCF Analysis for Grind Simulation

Synthetic TOR

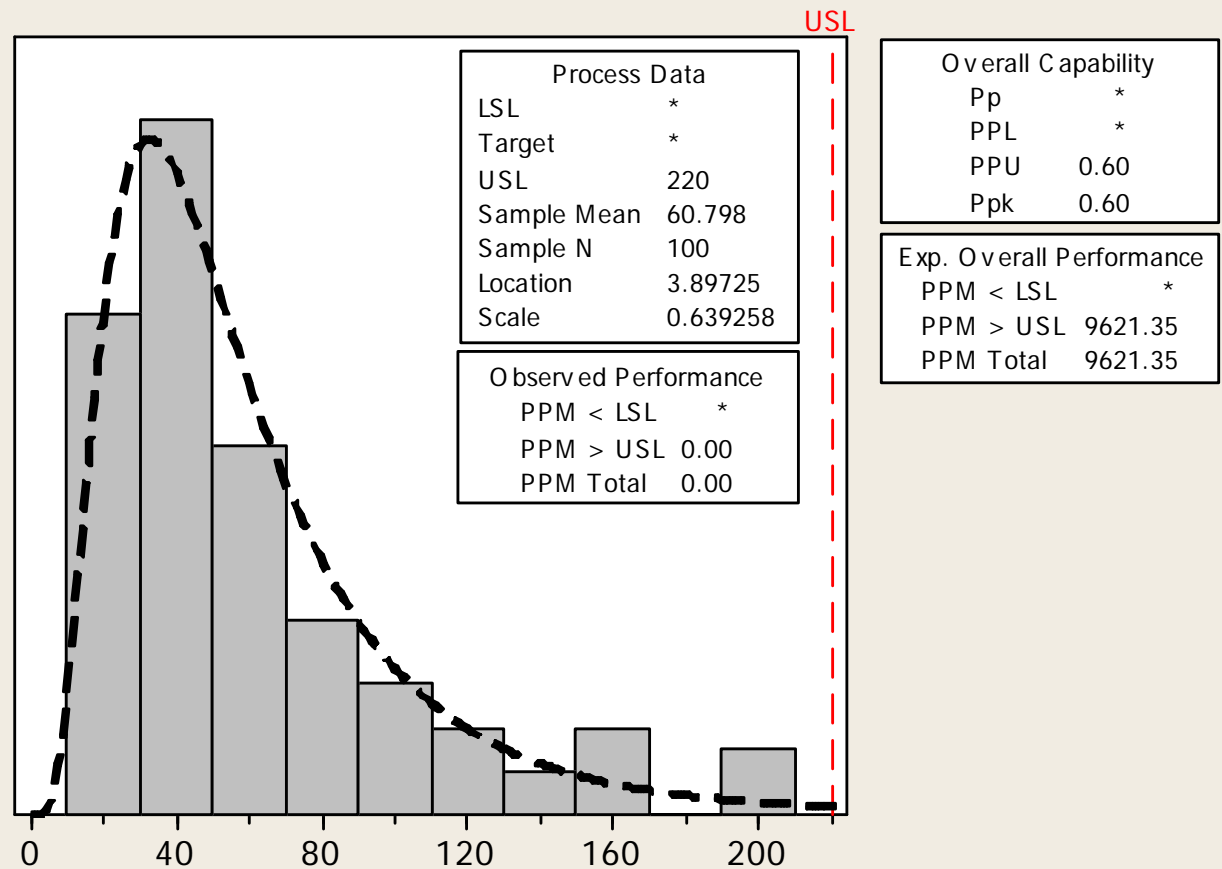
N = 100

Mean = 61 μm

- 220 μm Grind Target
- Based upon <1% of cracks having remainder after grind simulation

Process Capability of Depth (μm) -Synthetic TOR

Calculations Based on Lognormal Distribution Model



RCF Analysis for Grind Simulation

Water Based TOR

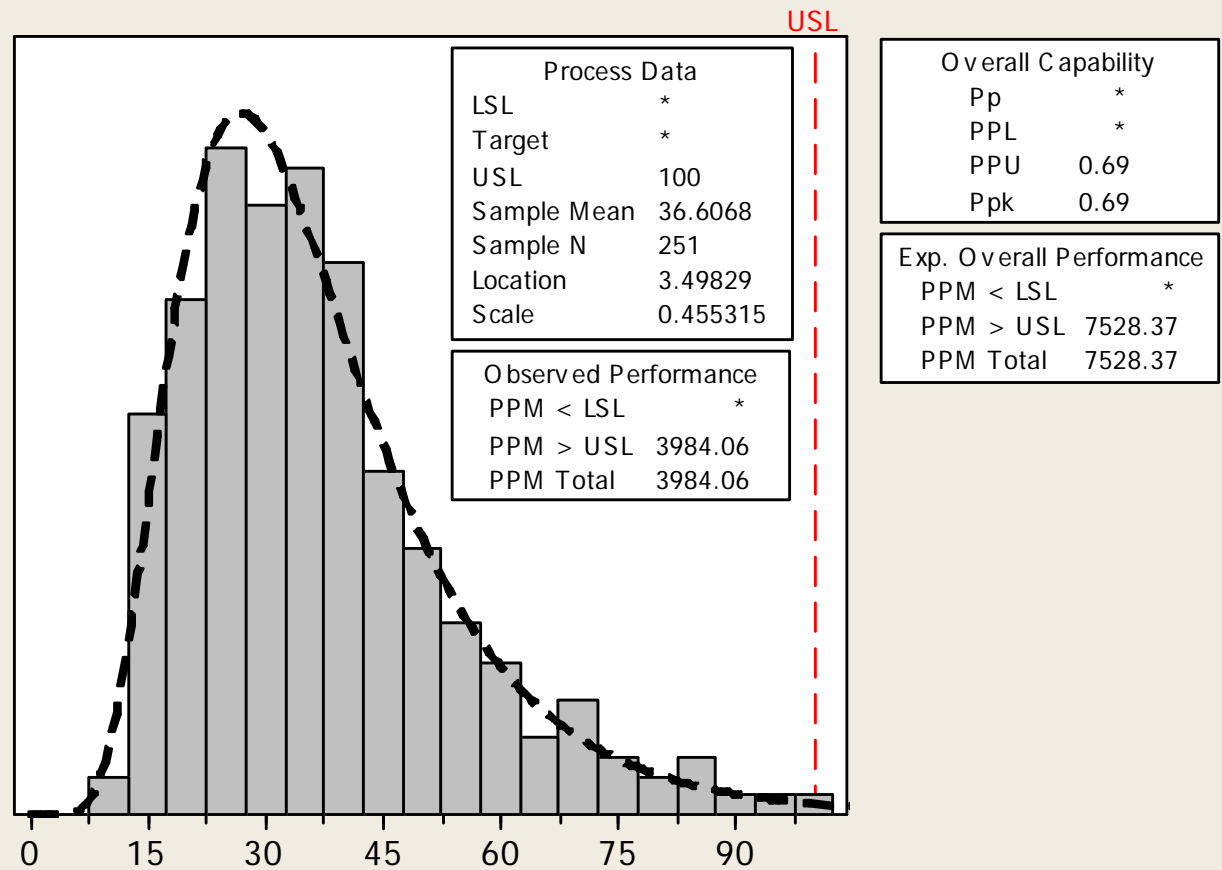
N = 251

Mean = 37 μm

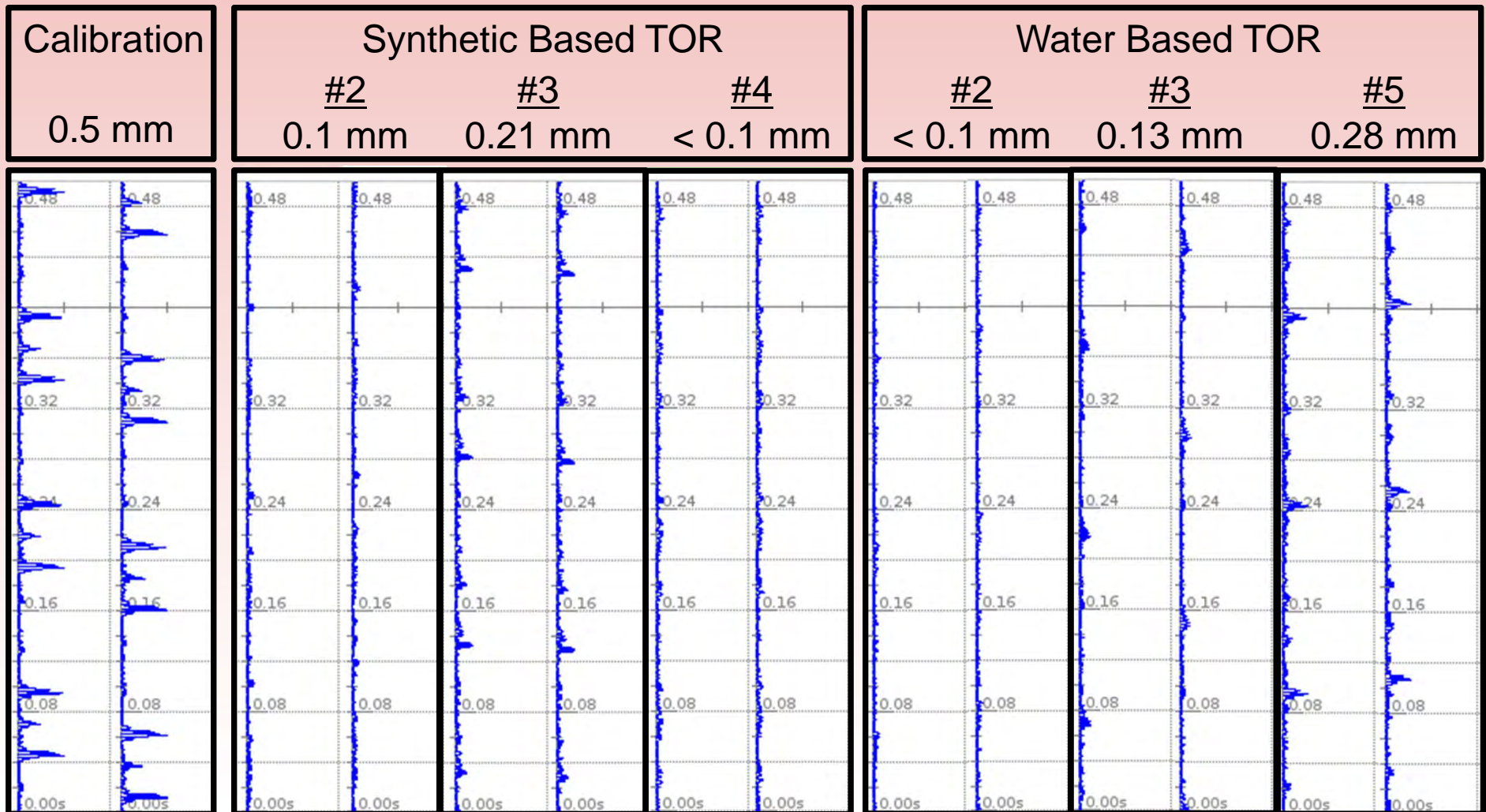
- 100 μm Grind Target
- Based upon <1% of cracks having remainder after grind simulation

Process Capability of Depth (um) - Water Based TOR

Calculations Based on Lognormal Distribution Model



Phase 2: Testing After Grinding



Phase 2: Testing After Grinding

Based upon eddy current analysis of the 6 grind-simulation samples

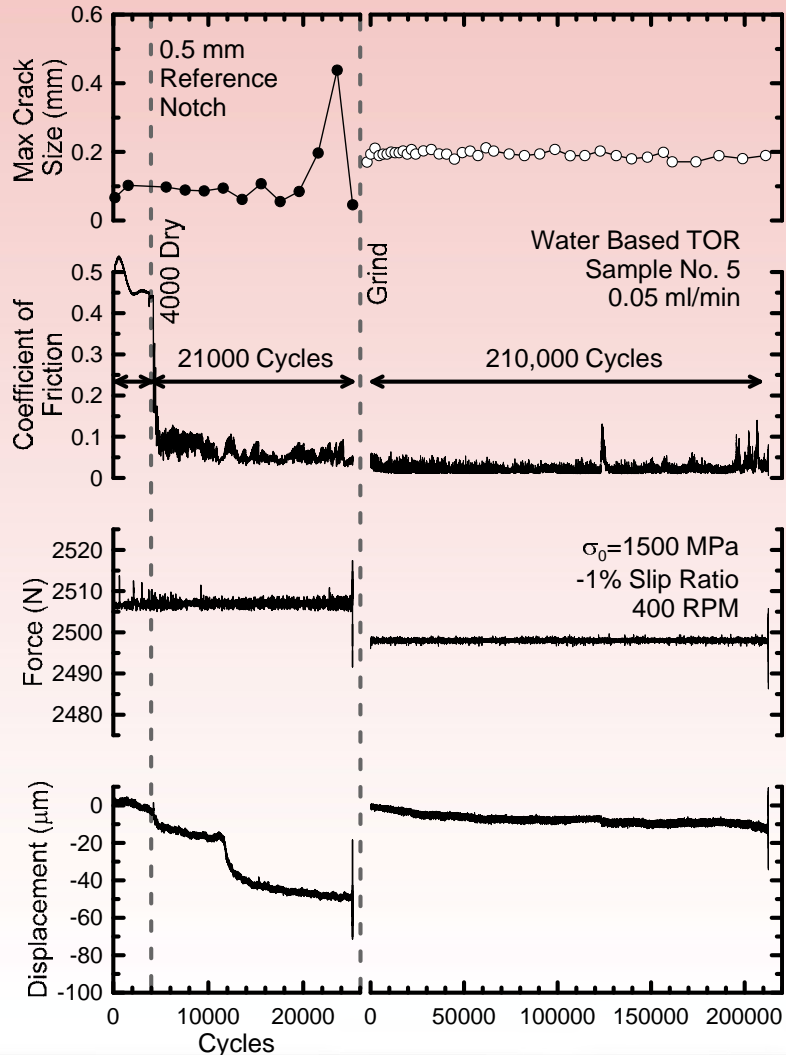
For each TOR material, a sample set representing 3 residual crack conditions were obtained after grind simulation

1. 1 disk with no measureable cracks (< 0.1 mm)
2. 1 disk with trace amounts of cracks (~ 0.1 mm)
3. 1 disk of very detectable cracks (> 0.2 mm)

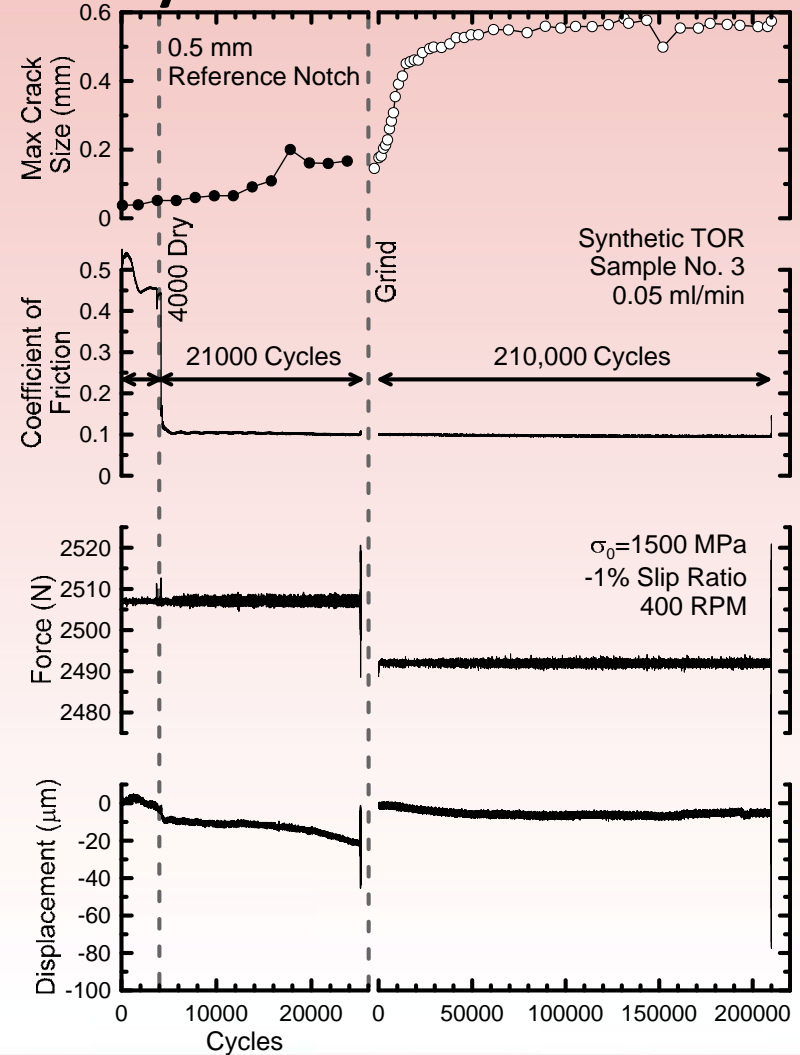


TOR Comparison

Water Based TOR – 0.28 mm



Synthetic TOR – 0.21 mm



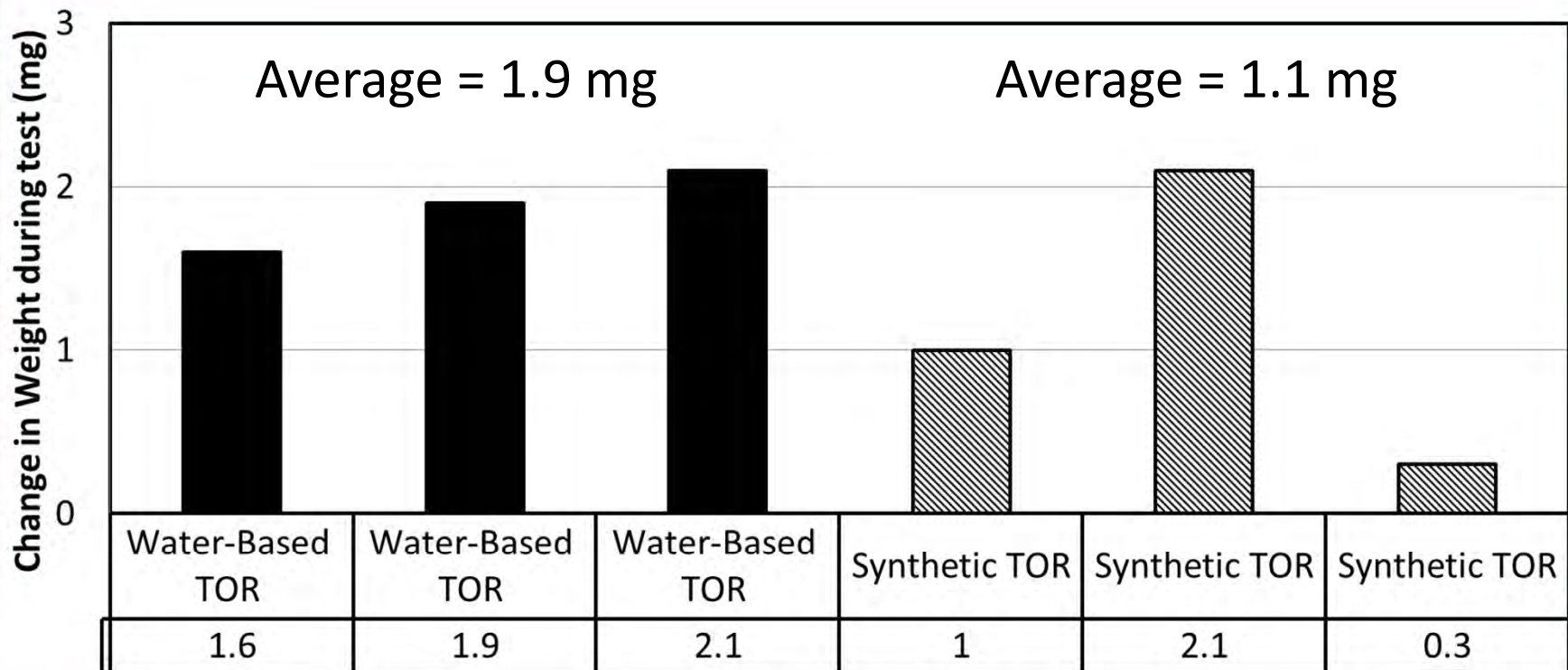
TOR Comparison

Water Based TOR

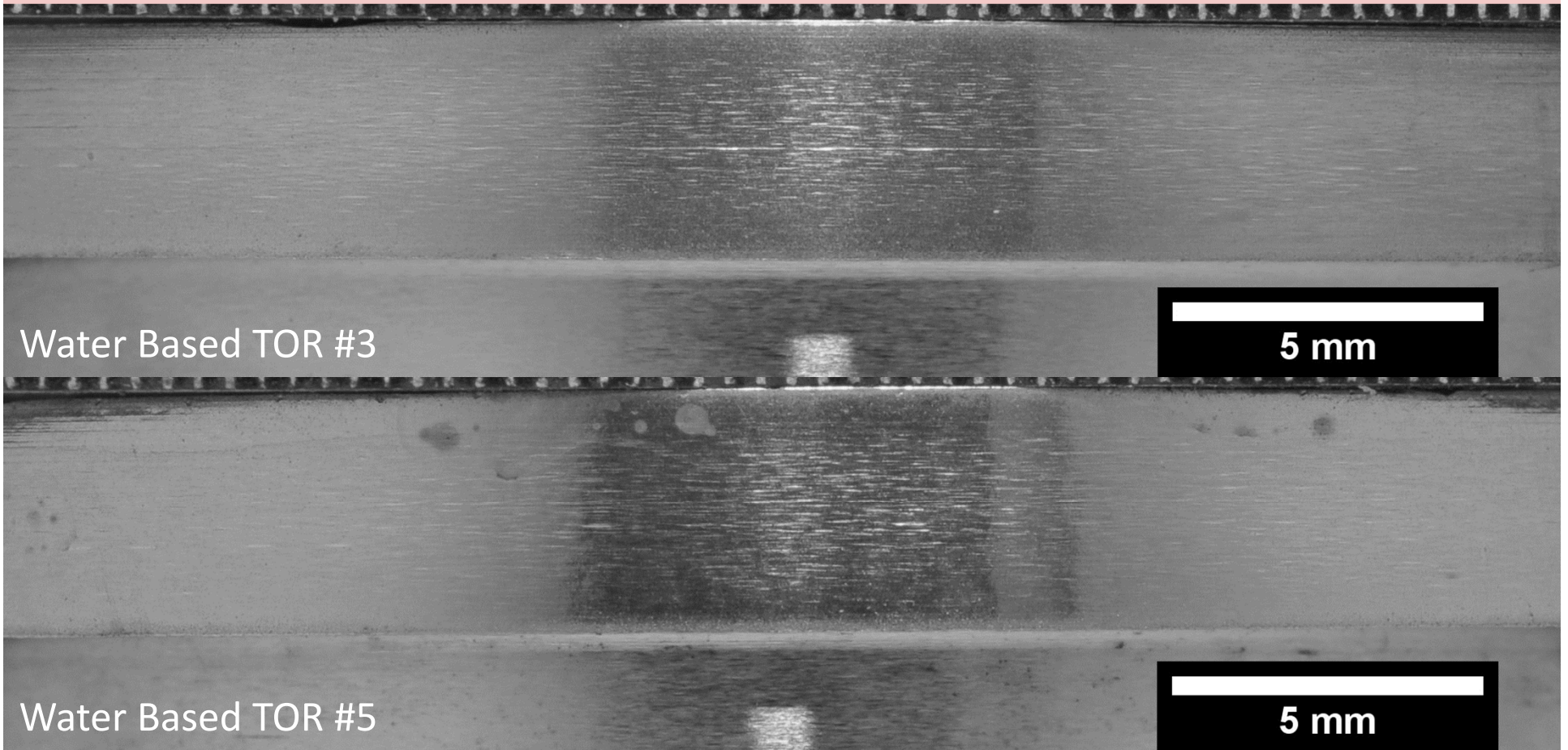
Part 1 – Average
93.2 mg / 25000 cycles

Synthetic TOR

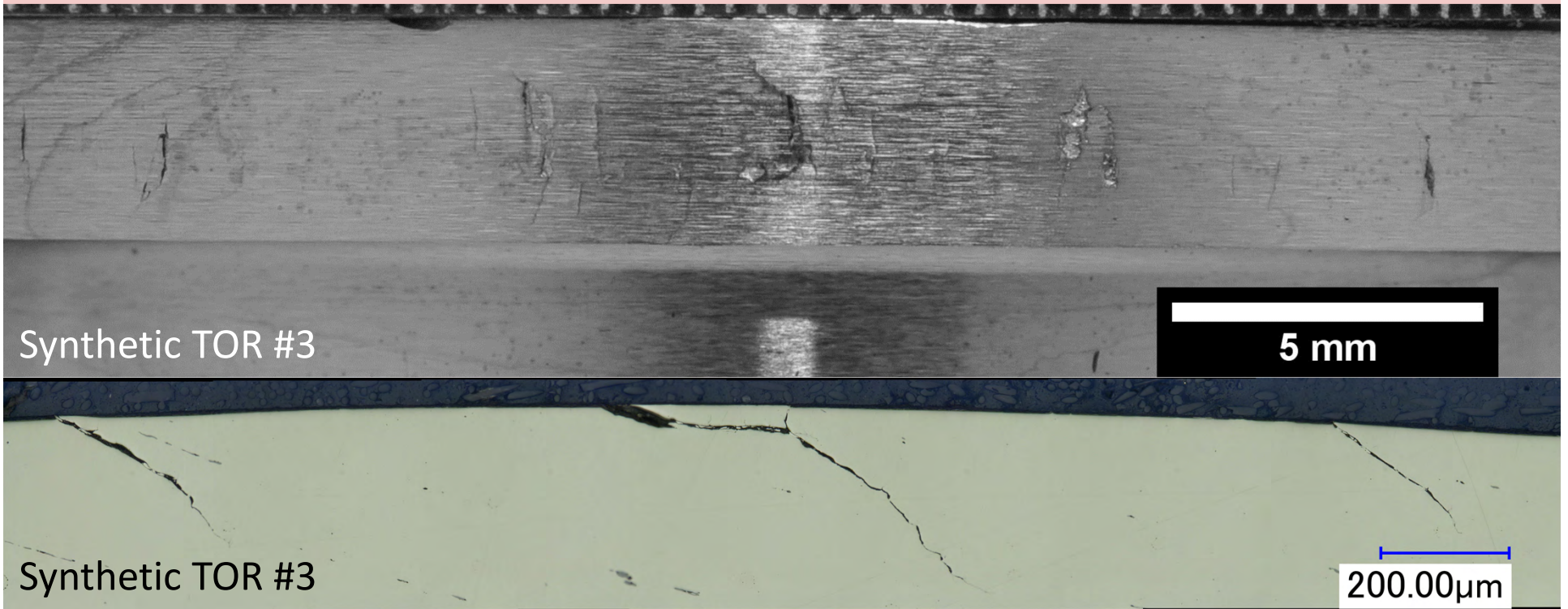
Part 1 – Average
55 mg / 25000 cycles



210K Lubricated



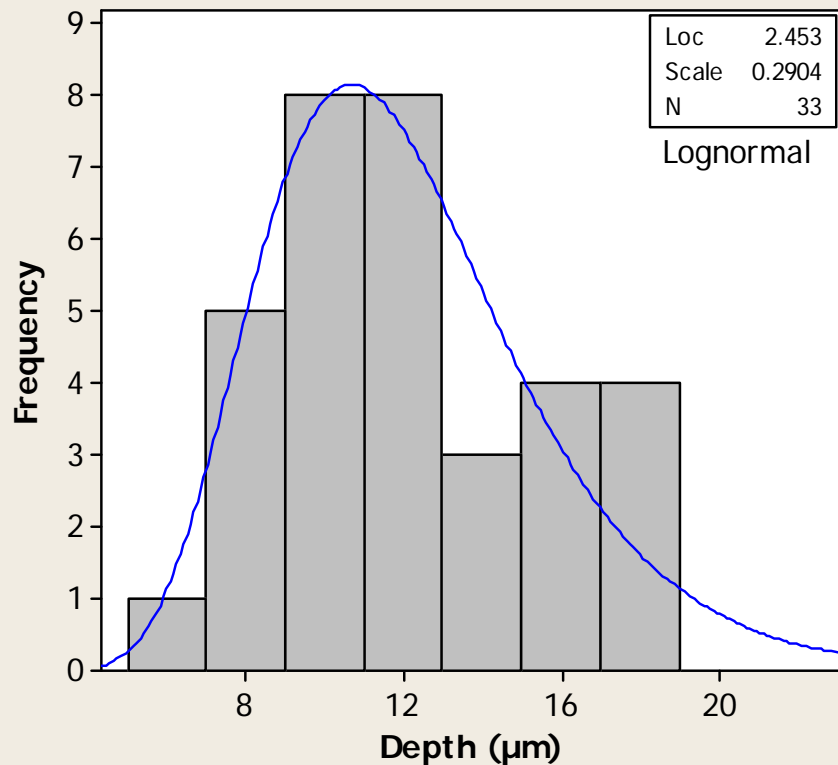
210K Lubricated



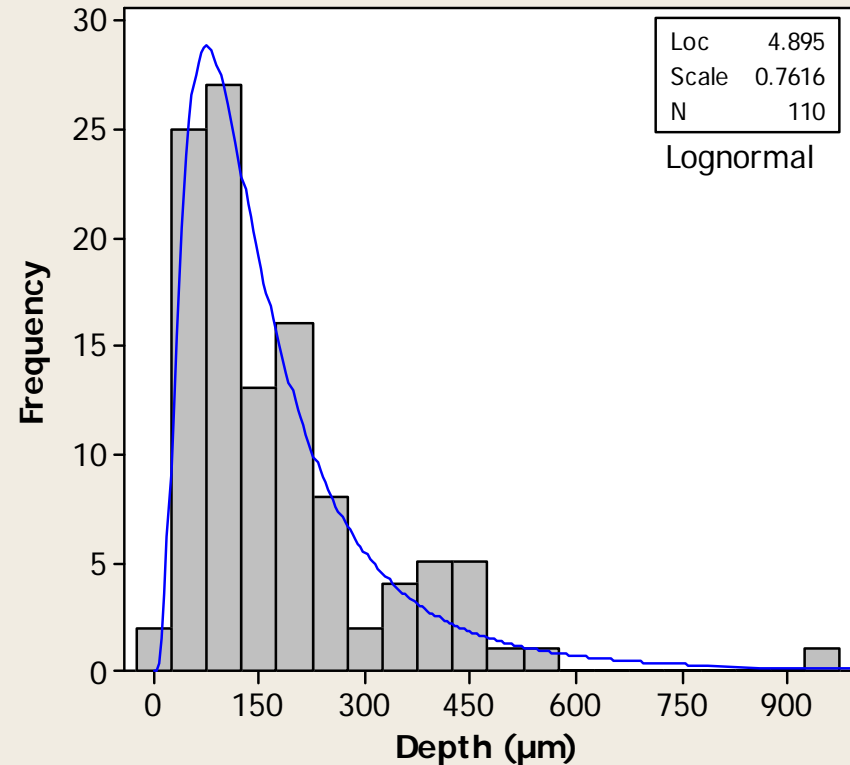
Crack Propagation

Crack population (combined, 3 samples each) show that the Synthetic TOR exhibits higher crack counts and larger depth values

Histogram of Depth (μm) - Water Based TOR



Histogram of Depth (μm) - Synthetic TOR



Considerations

Sub-sized Testing vs. Revenue Service Conditions

1. Stress field/microstructure scale differences (1:30th scale)
2. Friction coefficient not representative of revenue service
 - COF - Water Based TOR - **0.07** Pt. 1, **0.03** Pt. 2
 - COF - Synthetic TOR - **0.11** Pt. 1, **0.10** Pt. 2
 - Lubrication rates likely in excess of what would represent revenue service conditions (modification for future study)
3. Severe surface layer damage during dry break-in results in wide-flat deformation cracks
 - Surface appearance is that the material is “rolled” out over adjacent material
 - Representative of “dry” service conditions



Comparisons to Revenue Service

Axle load ~ 36 tons = 1 wheel pass

Approximate, but conservative, estimate of MGT

- 4,000 dry cycles = 0.14 MGT (< 2 days on 36 MGT line)
- 25,000 cycles = 0.9 MGT (9 days on 36 MGT line)
- 210,000 cycles = 7.5 MGT (75 days on 36 MGT line)

Both TOR materials have been in use in revenue service

- Long term, continuous usage
- Synthetic TOR ~ 6 years in ~50 MGT line
- Water Based TOR ~ 7 years in ~70 MGT line
- No manifestation of damage as shown in the disc-on-disc testing
 - Grinding practice was in place



Conclusions

The Effects of Grinding

1. The grind simulation samples do have significantly less crack propagation than the original study at 10X cycles.
2. If there is residual RCF after grinding, that could be the area of growth depending on the Friction Modifier used.
3. When using an HH material there seems to be a max depth of crack propagation.
4. The HH rail did not have the severe RCF propagation as that exhibited by the rails used in the previous study (WRI 2014)
5. If FM equipment in the field is OOS, it is important to have systems return to service in a timely manner to minimize crack initialization.



**Jim
Thompson**

**Greg
Garcia**

**Nick
Dryer**

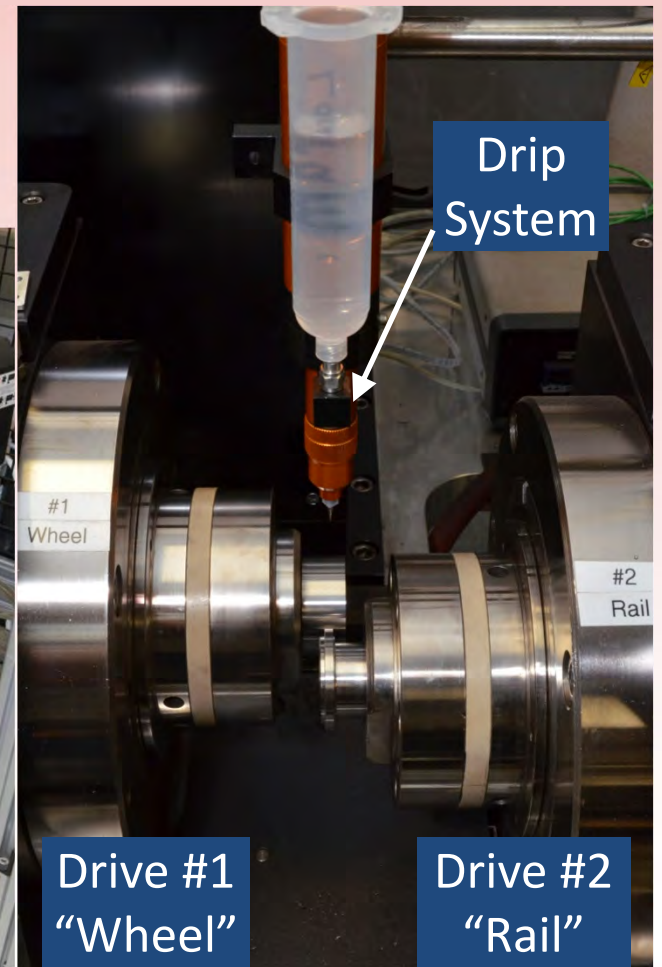
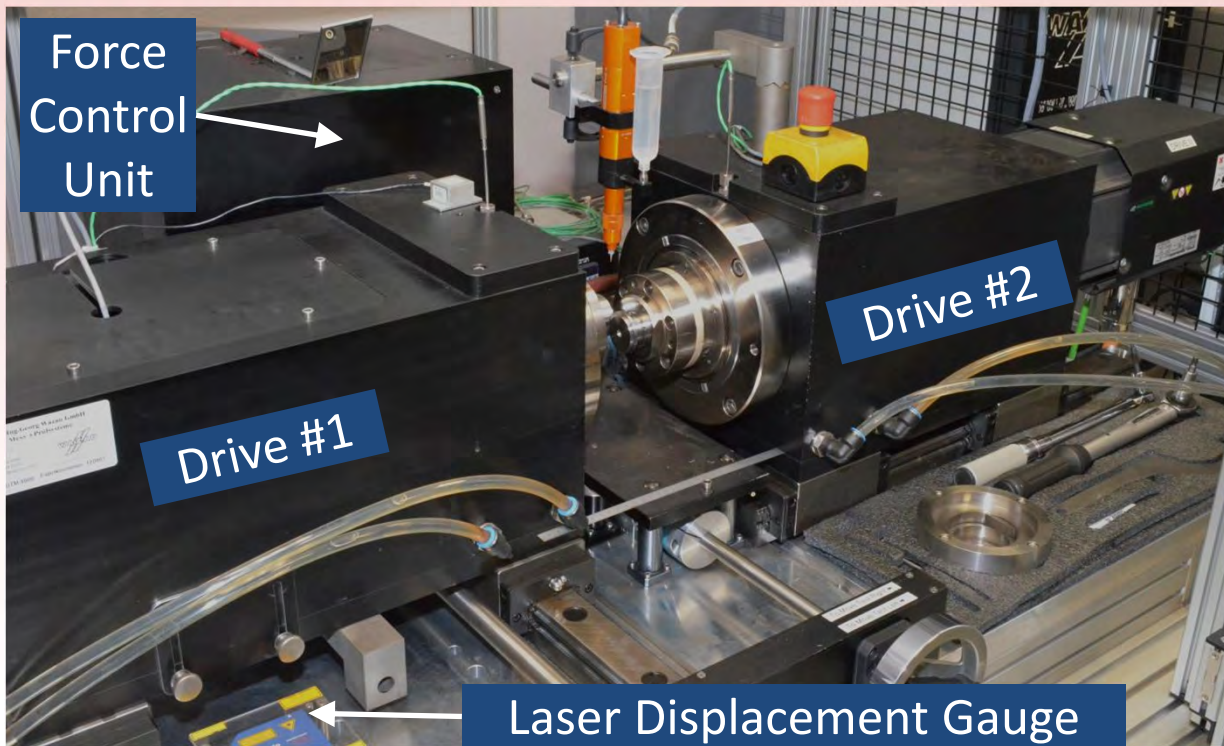
**Greg
Lehnhoff**

**Mark
Richards**



Wazau UTM 5000

- 5000 N Force Capacity
- 50 N-m Torque Capacity
- 3000 RPM, independent drive spindles
- *in-situ* eddy current
- Lubrication “drip” system - flow rate control



RCF Initialization

- Rolling Contact Fatigue

- Reproduce the conditions of Hardwick at University of Sheffield
- Sample geometry and twin disc machine parameters are modified to allow increased contact pressures and rotational speed
- Close control of slip ratio (creepage) and applied loads (contact pressure)
- Lubrication is applied after a dry cycle break in period

- 1500 MPa contact pressure
where:

$$P_{max} = \frac{2F}{\pi aL}$$

- F is the Force set point
- a is contact patch dimension
- L is the contact length

- -1% slip ratio (i.e. $V_{Wheel} > V_{Rail}$)
where:

$$\frac{Slip\ Ratio}{100\%} = \frac{2 \times (V_{Rail} - V_{Wheel})}{V_{Rail} + V_{Wheel}}$$

- V_{Rail} is the rail surface velocity
- V_{Wheel} is the wheel surface velocity



Following Pages (36-40 point)

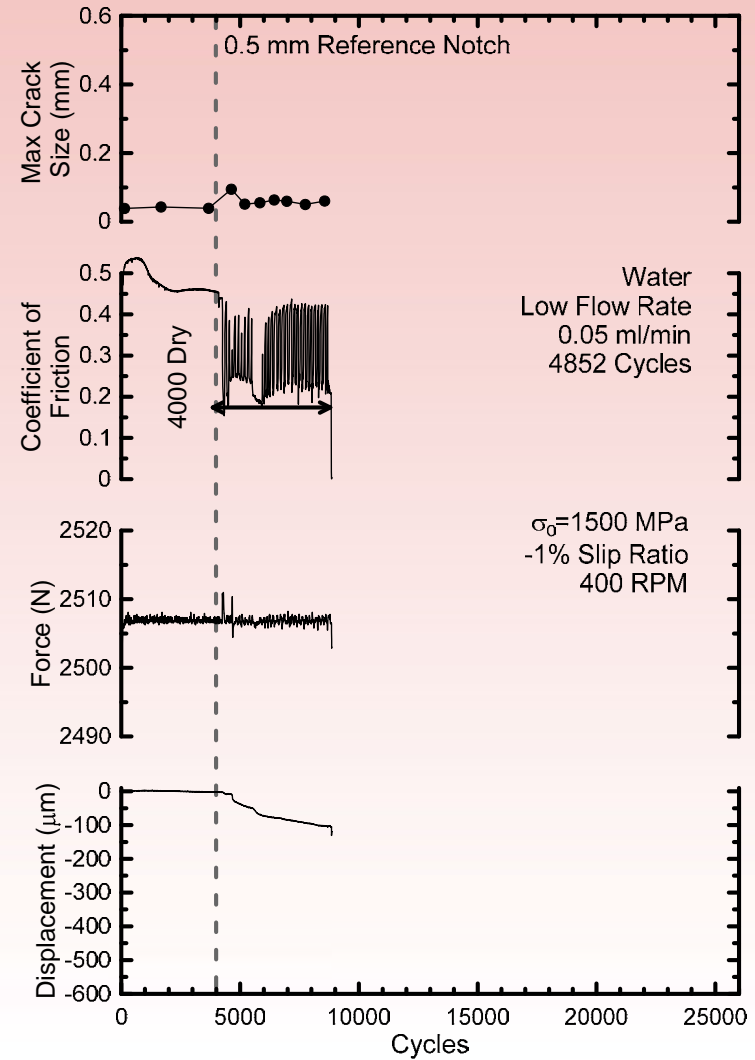
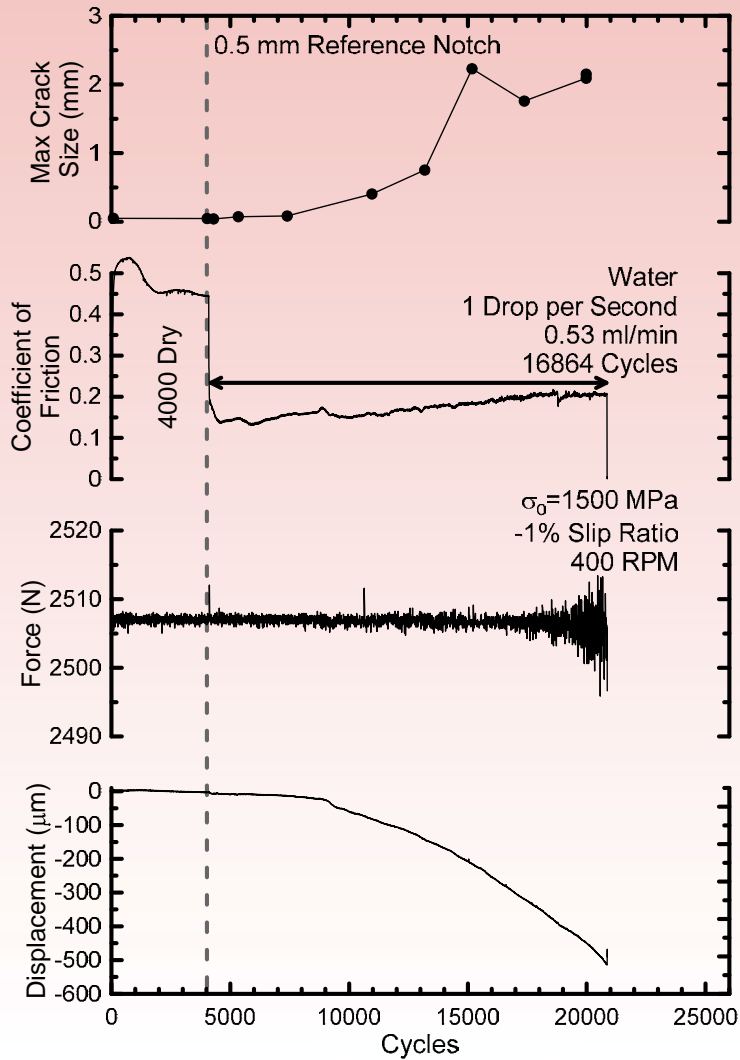
Subhead (28 point)

1. This is a sample of 24 point text in black.
2. This is a sample of 24 point text in black.
3. This is a sample of 24 point text in black.
4. This is a sample of 24 point text in black.

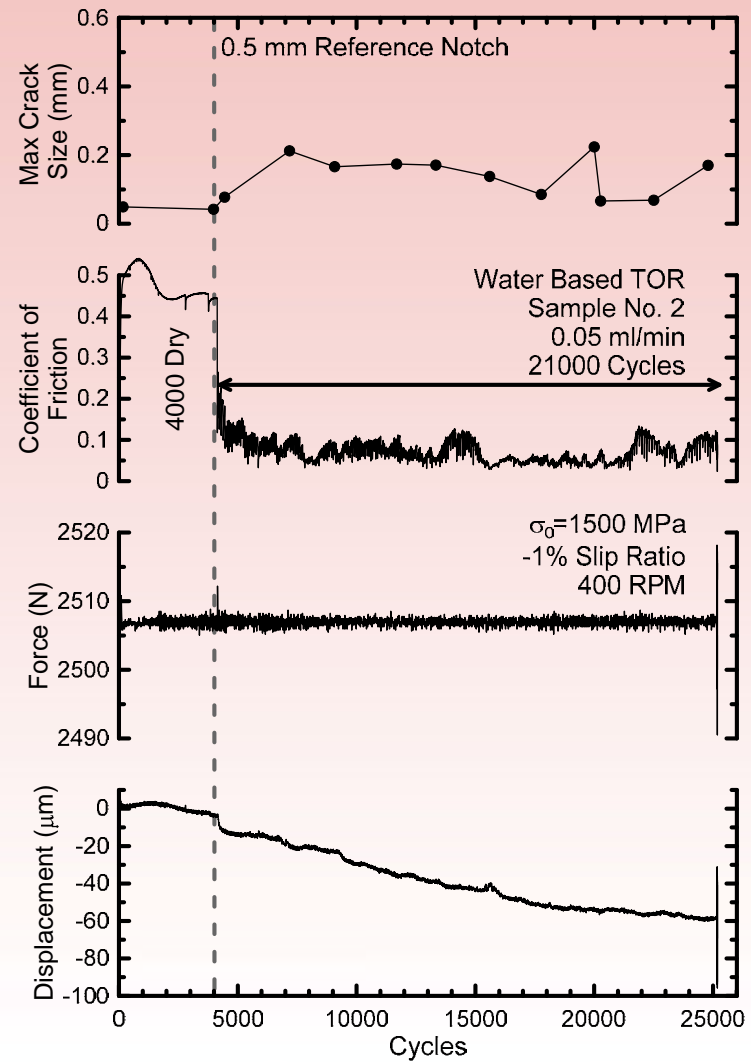
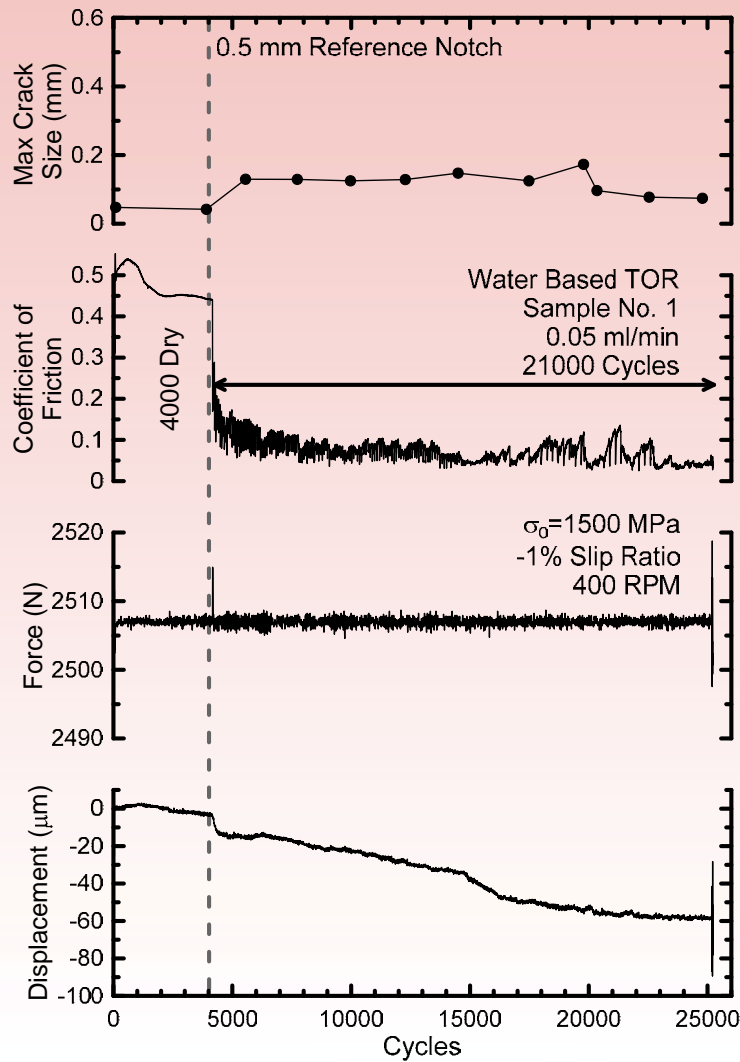
This is the minimum size text
(20 point) for readability



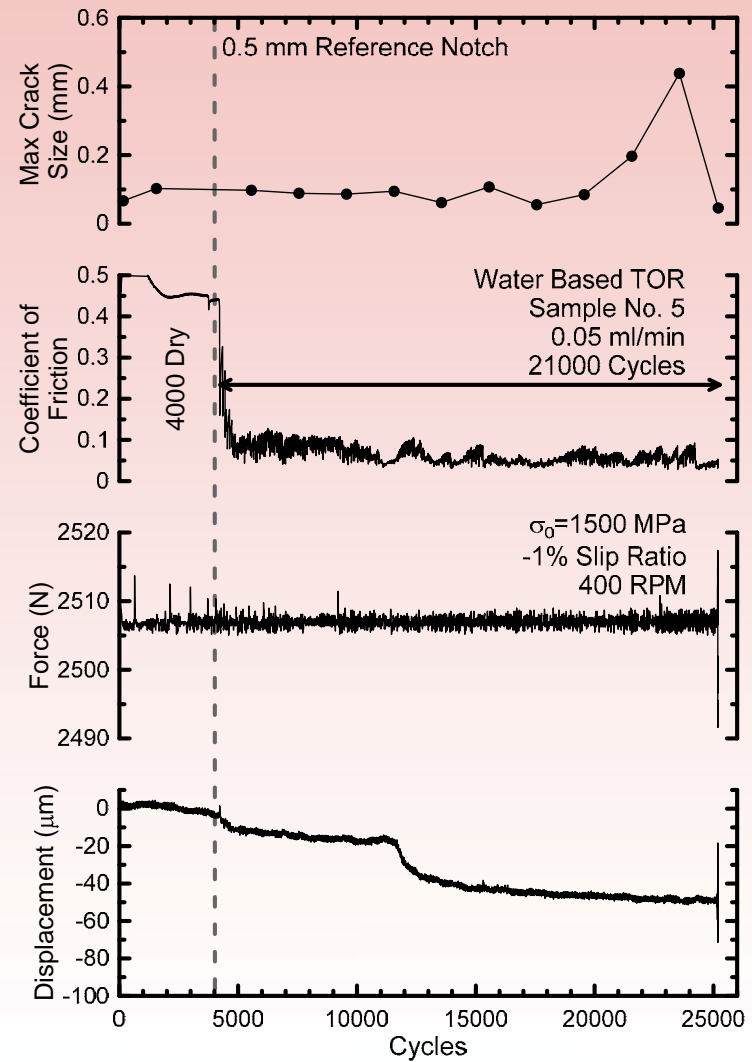
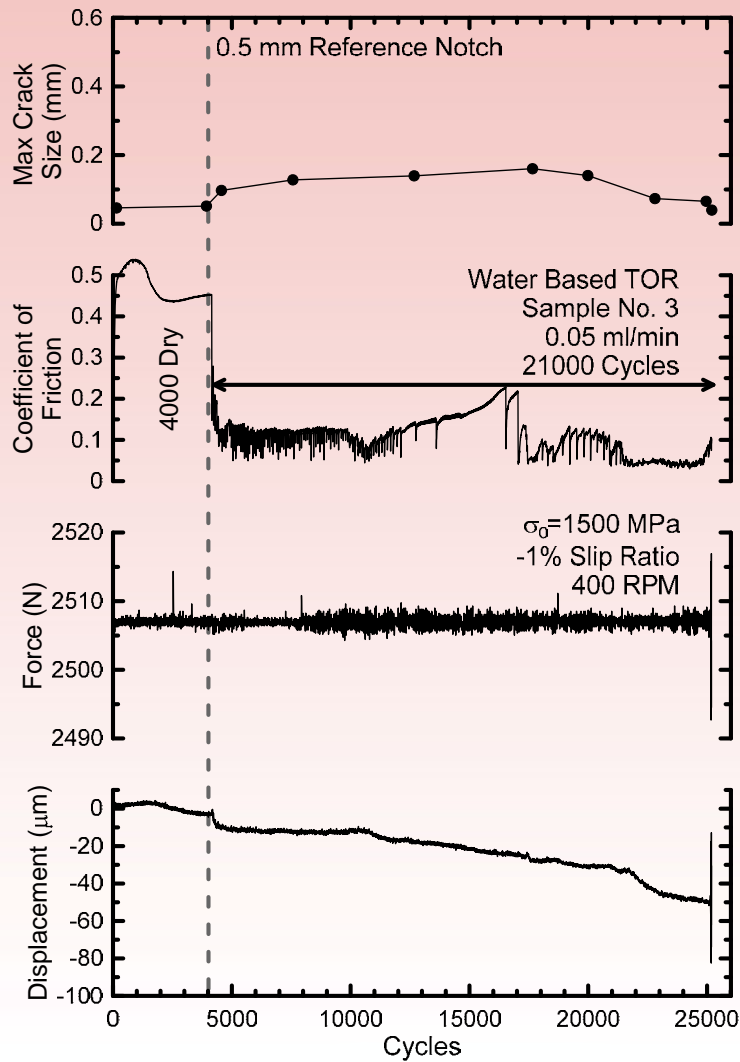
Water Part 1



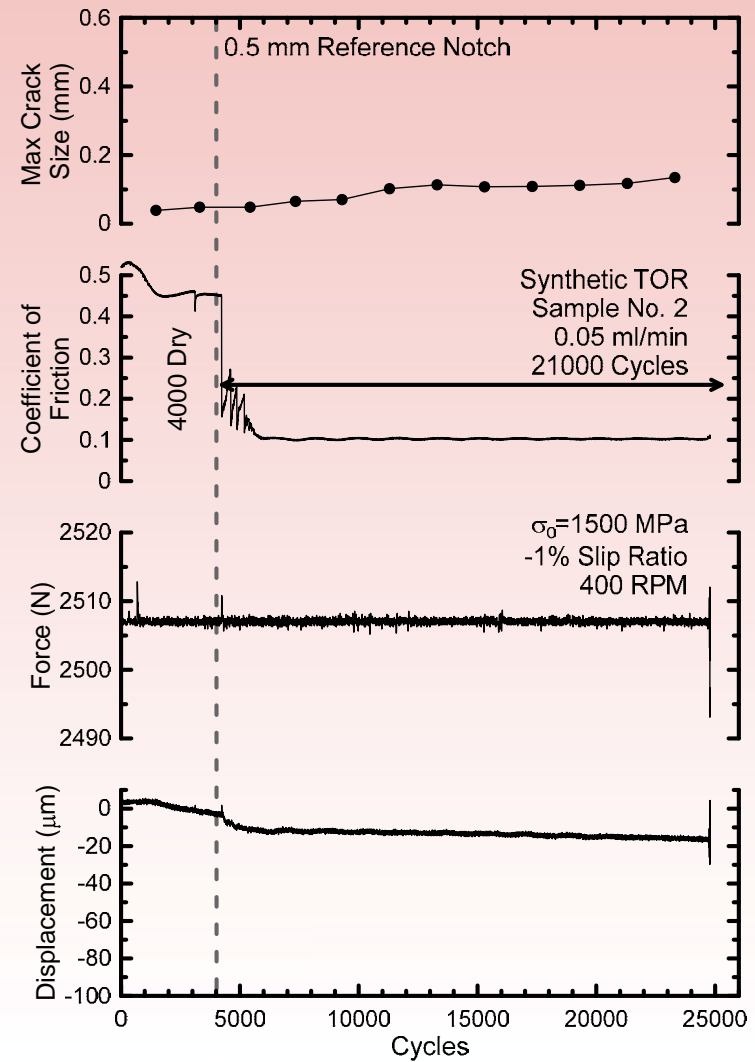
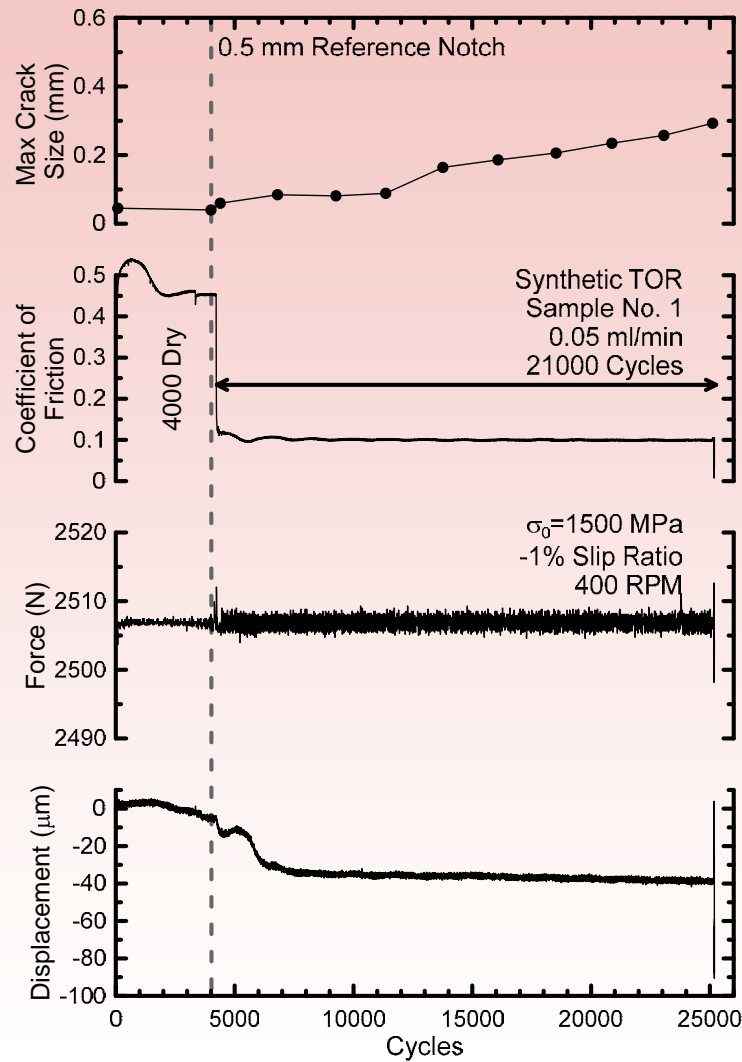
Water Based TOR Part 1



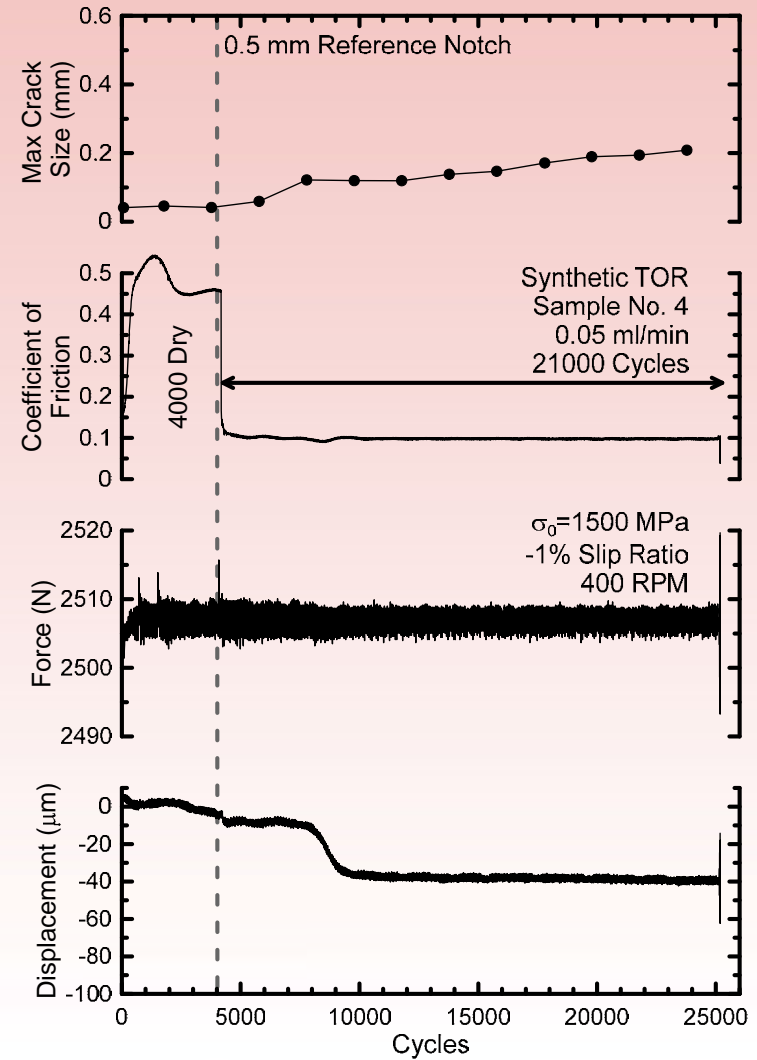
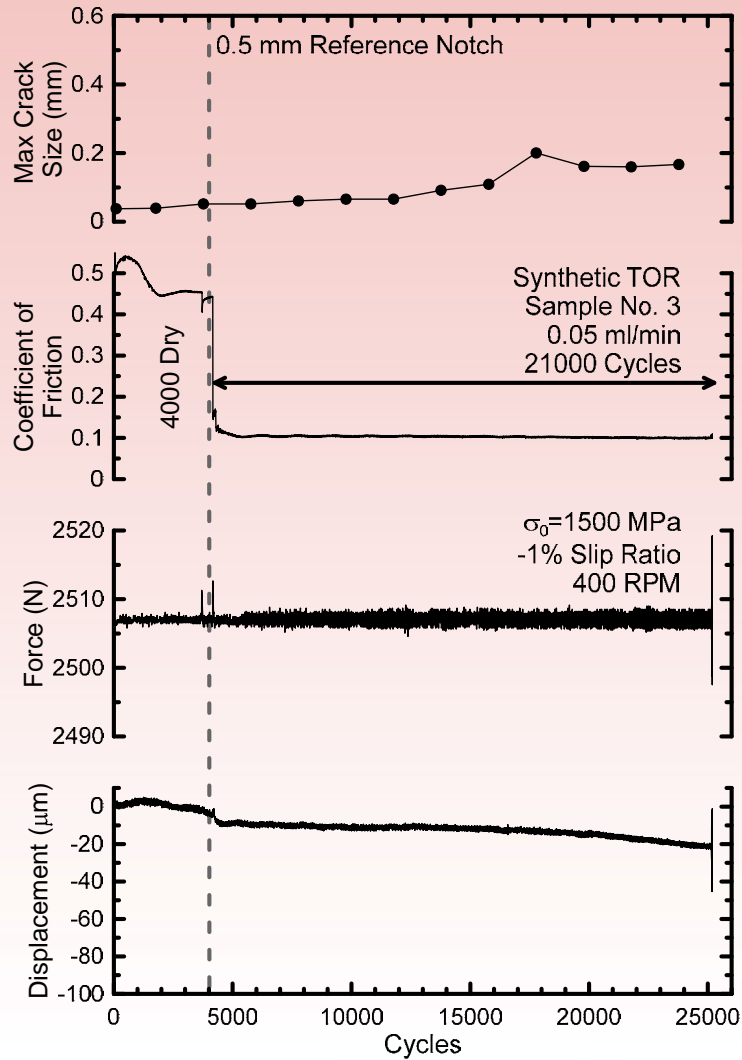
Water Based TOR Part 1



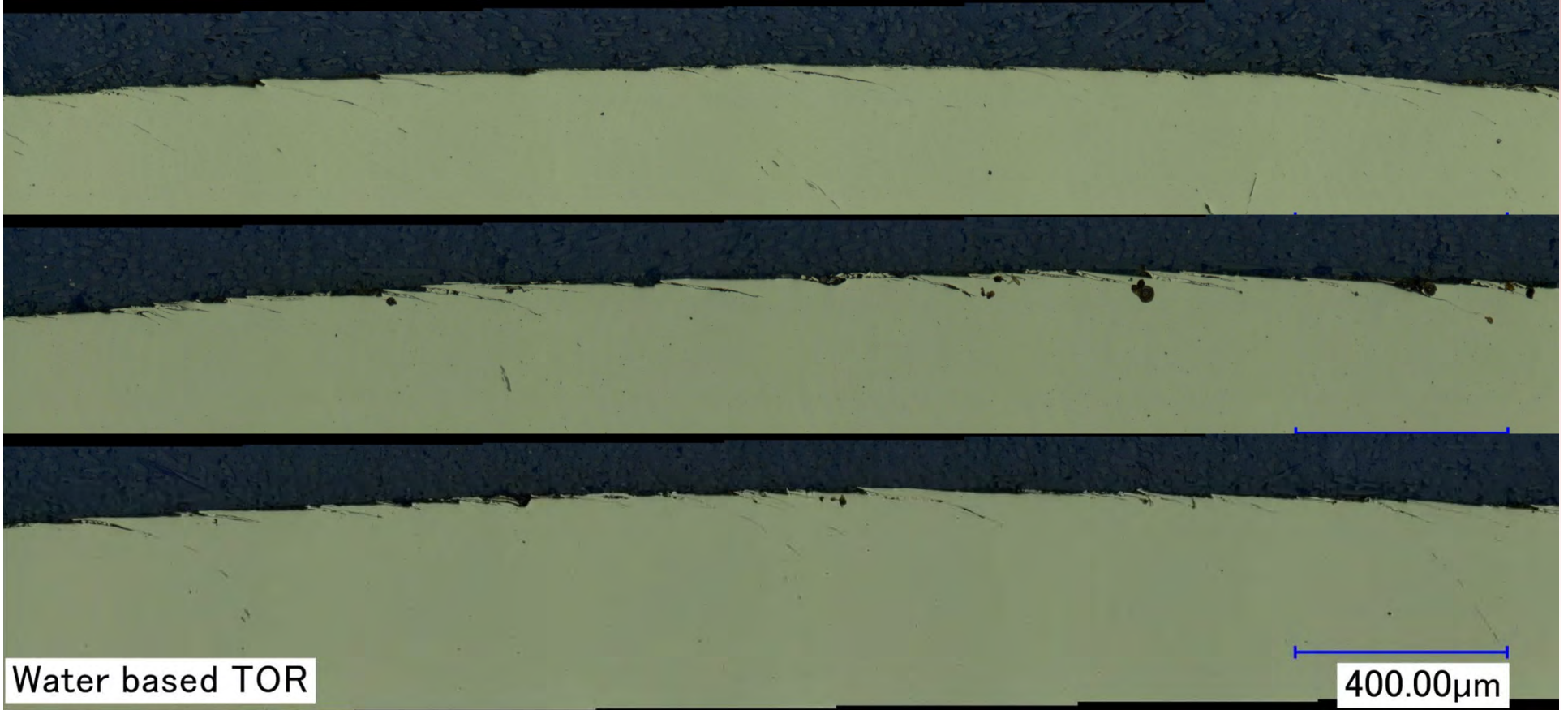
Synthetic TOR Part 1



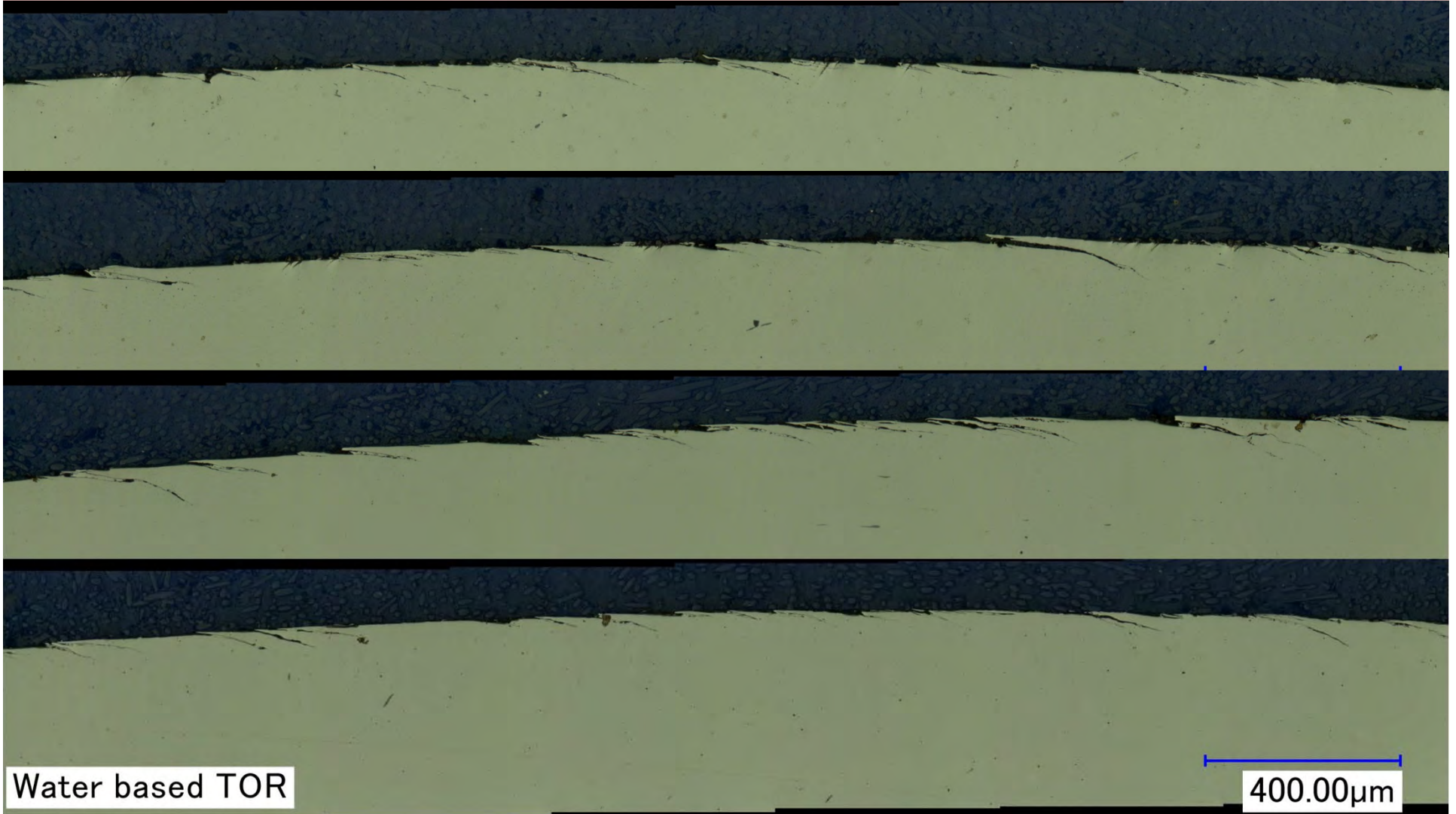
Synthetic TOR Part 1



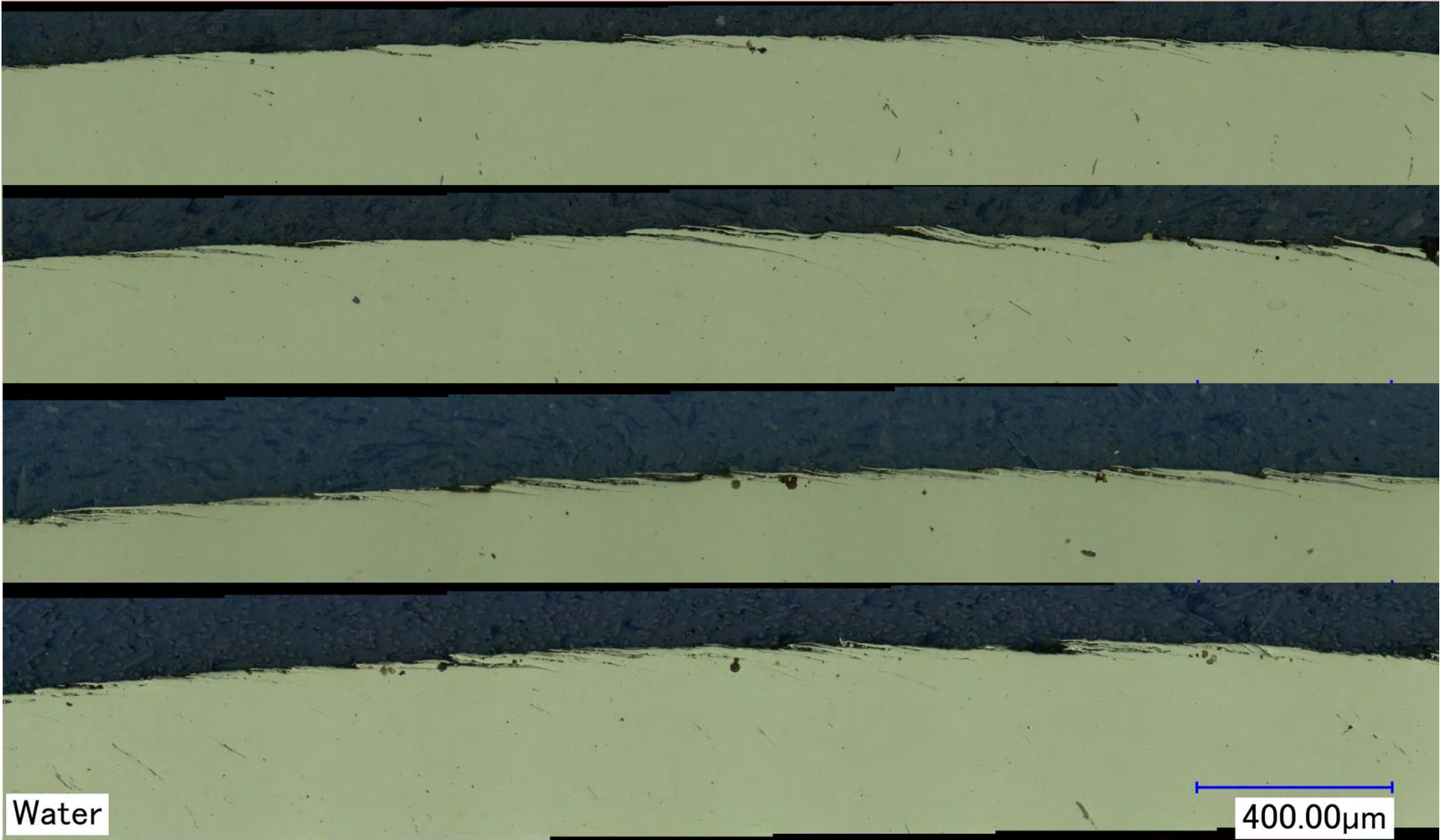
Water Based TOR Part 1



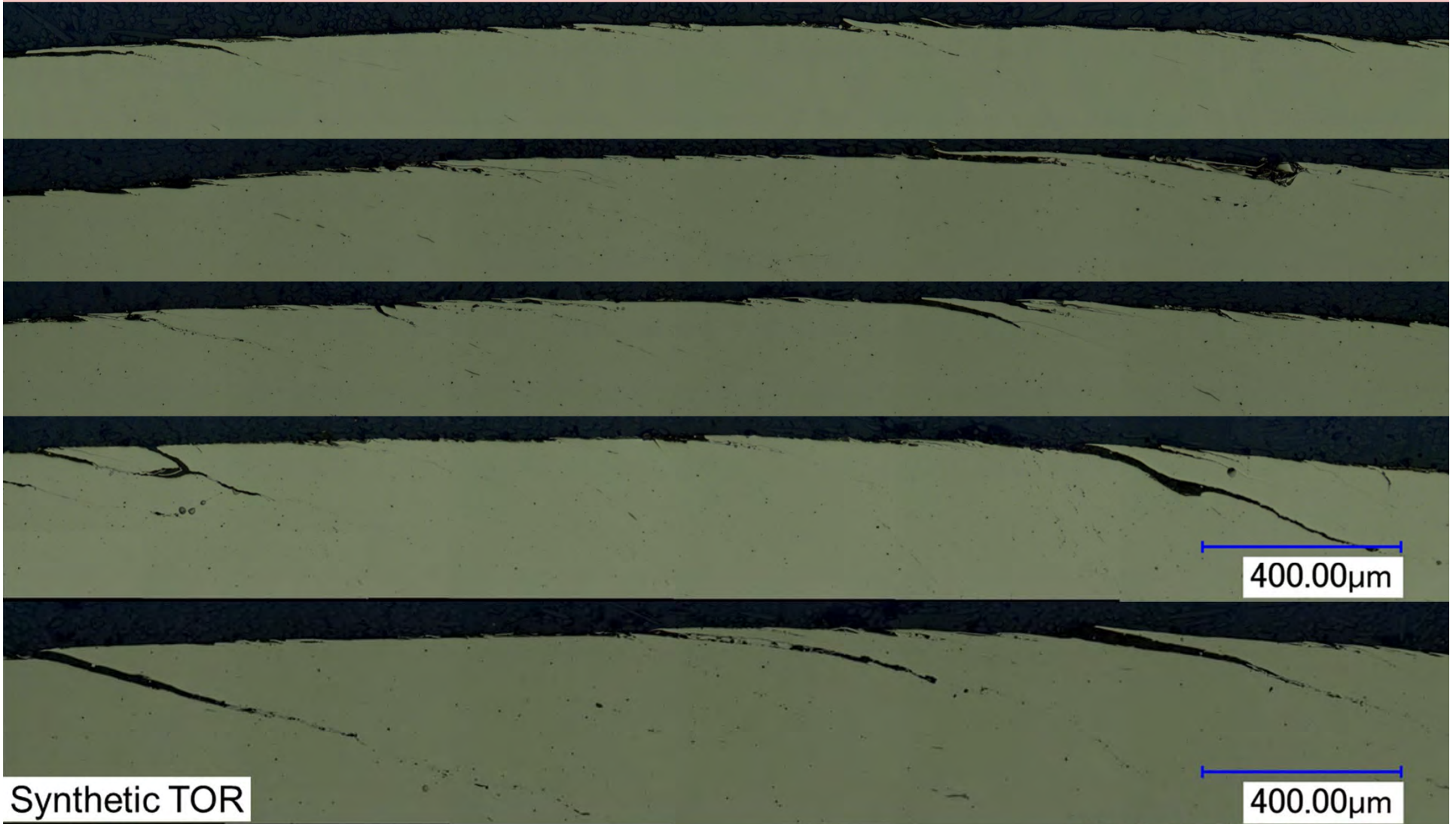
Water Based TOR



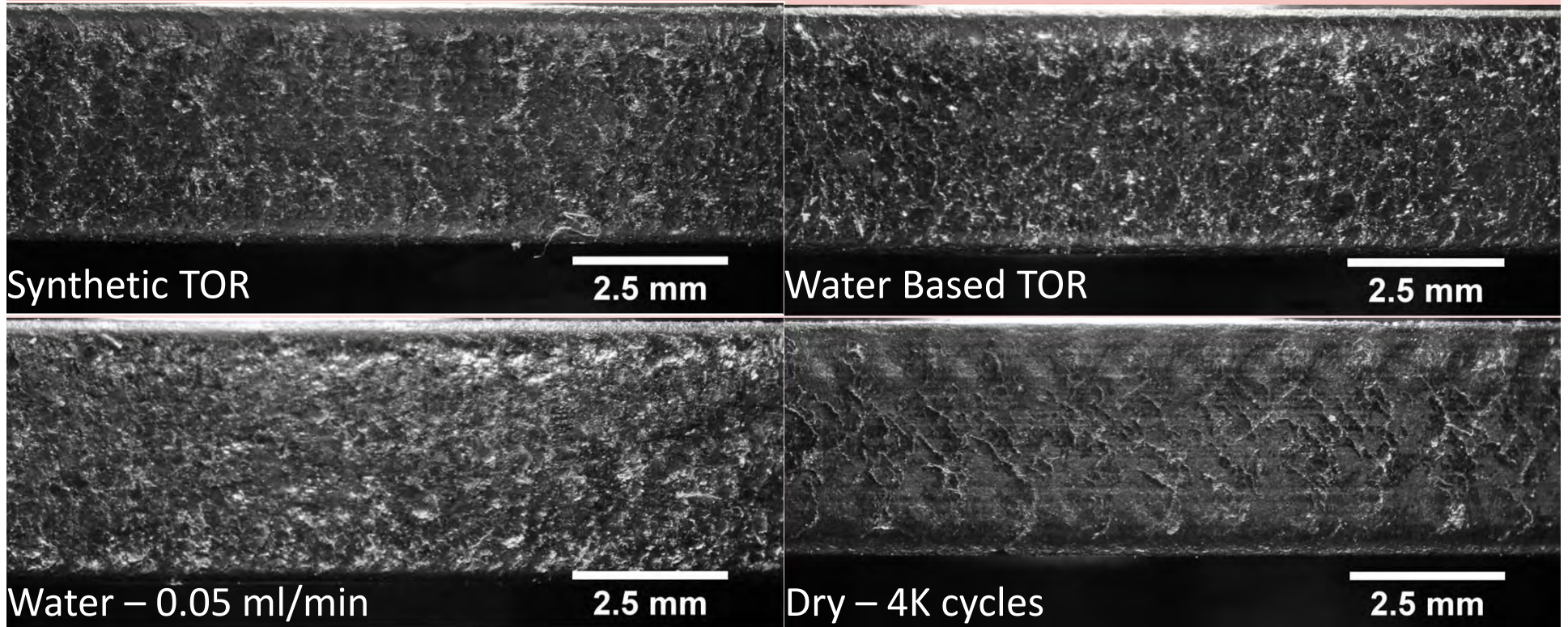
Water – 0.05 ml/min



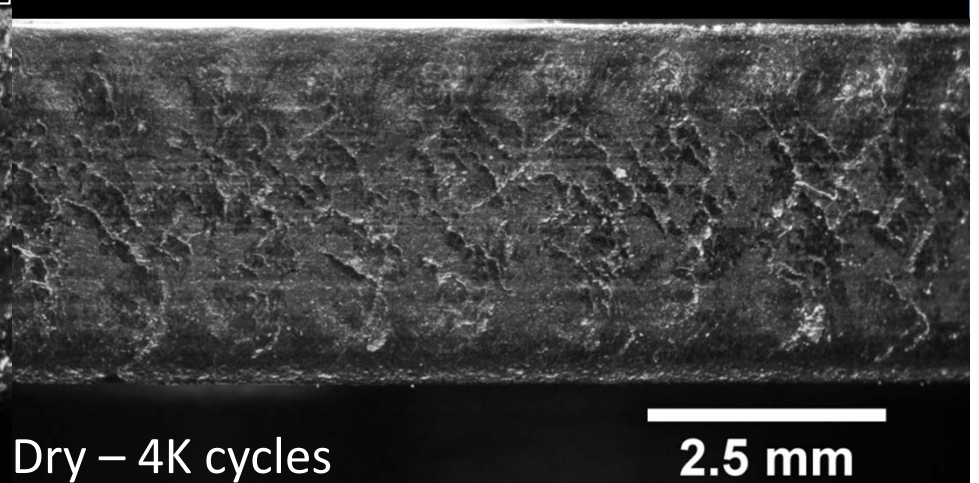
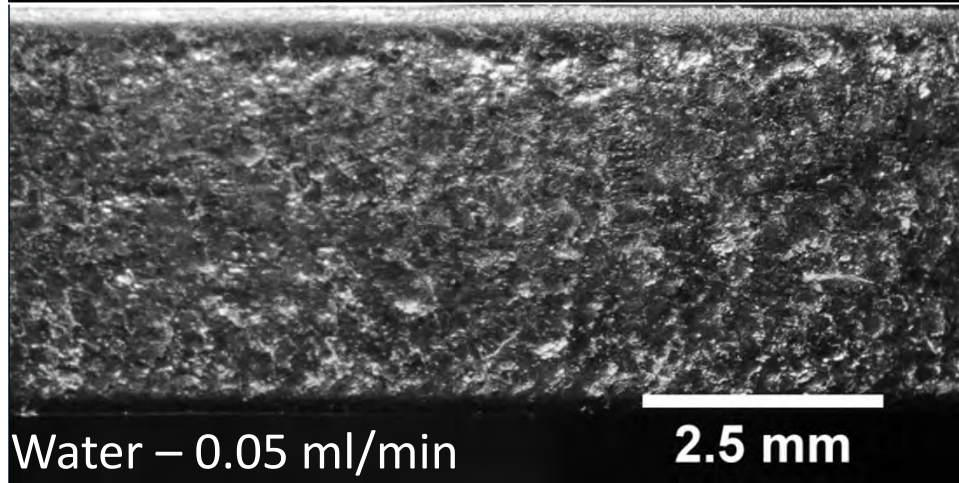
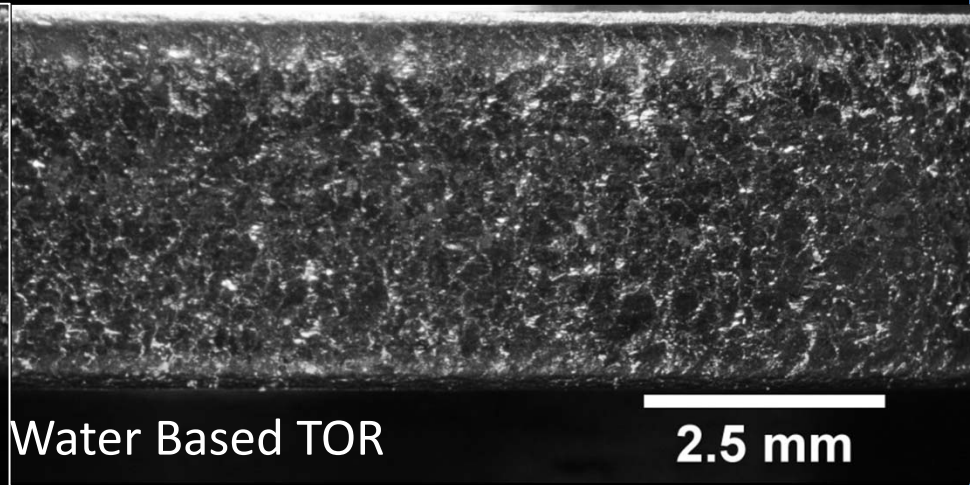
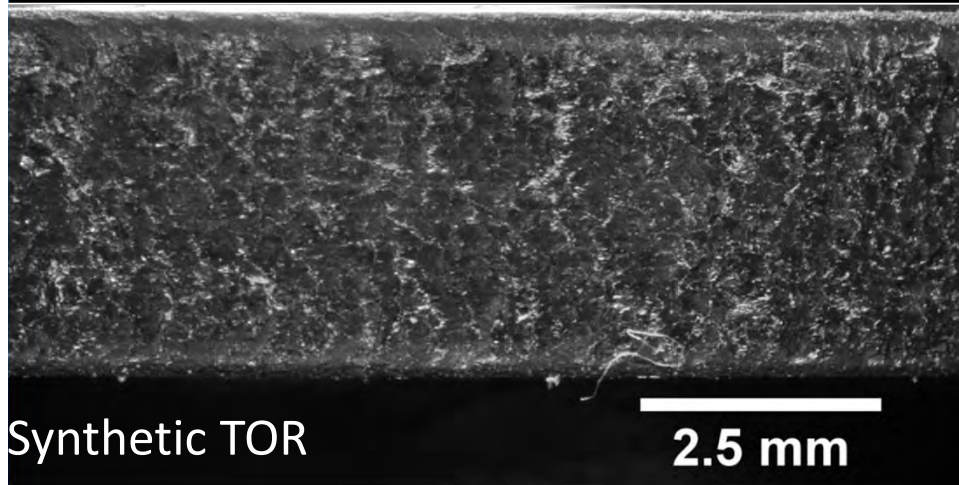
Synthetic TOR



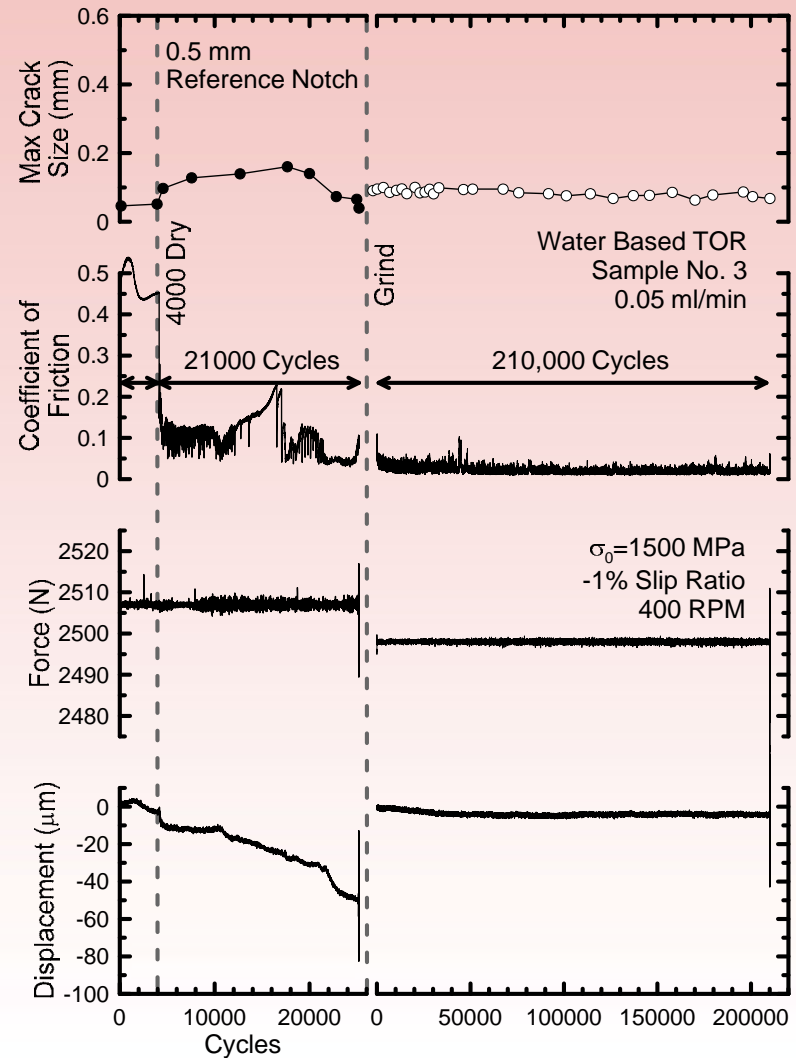
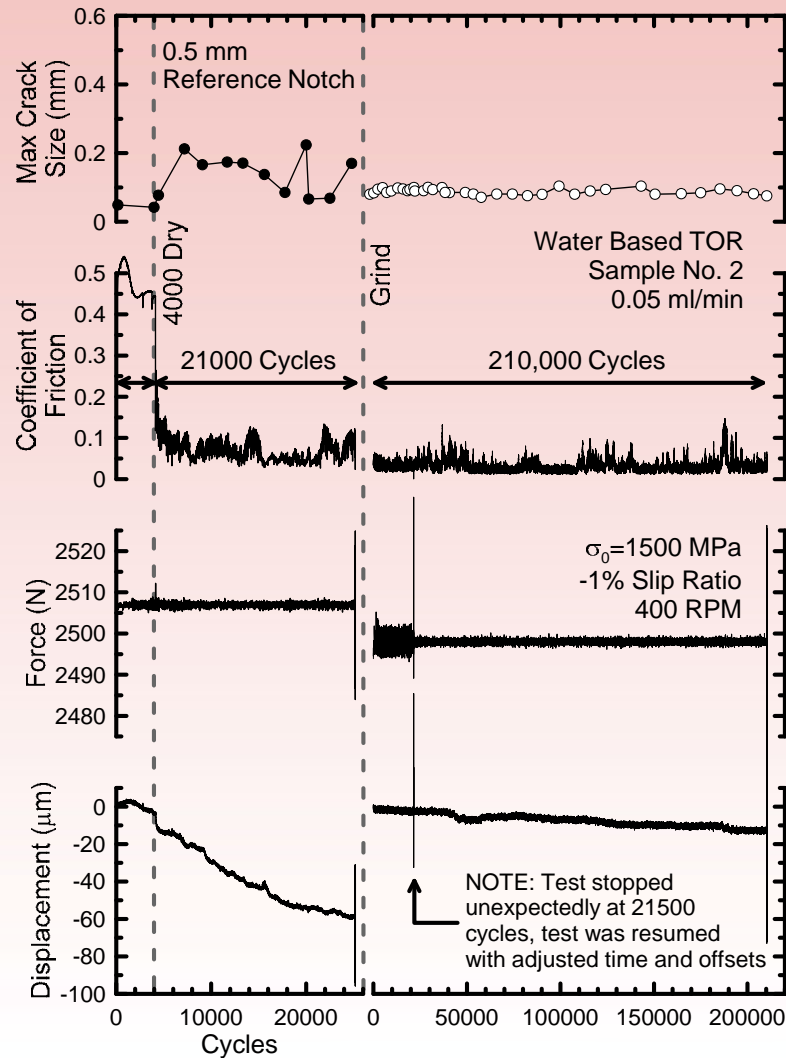
Dry, 21k Lubricated



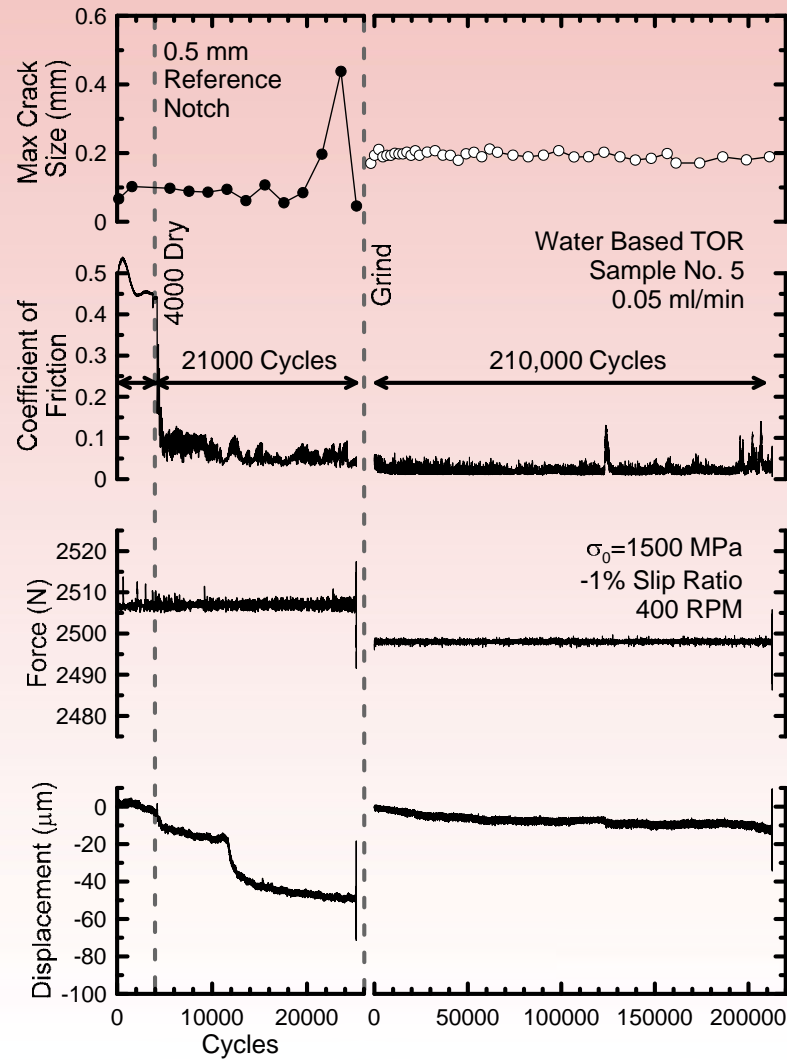
4k Dry – 21k Lubricated



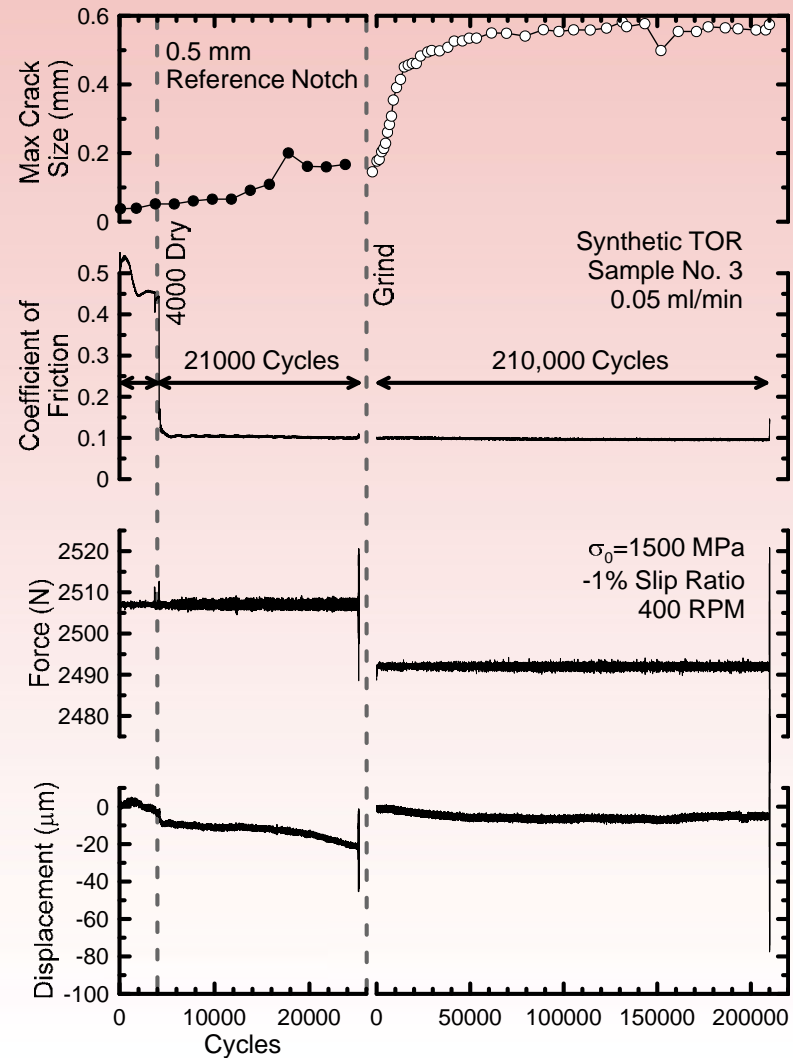
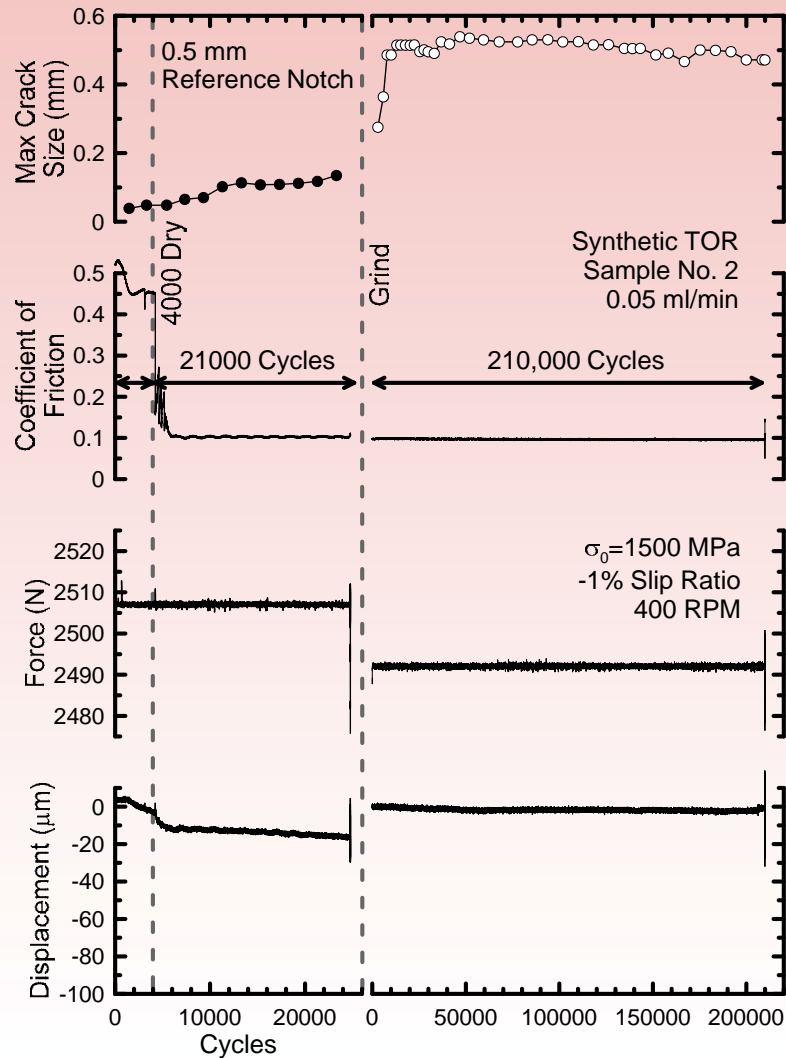
Water Based TOR Part 2



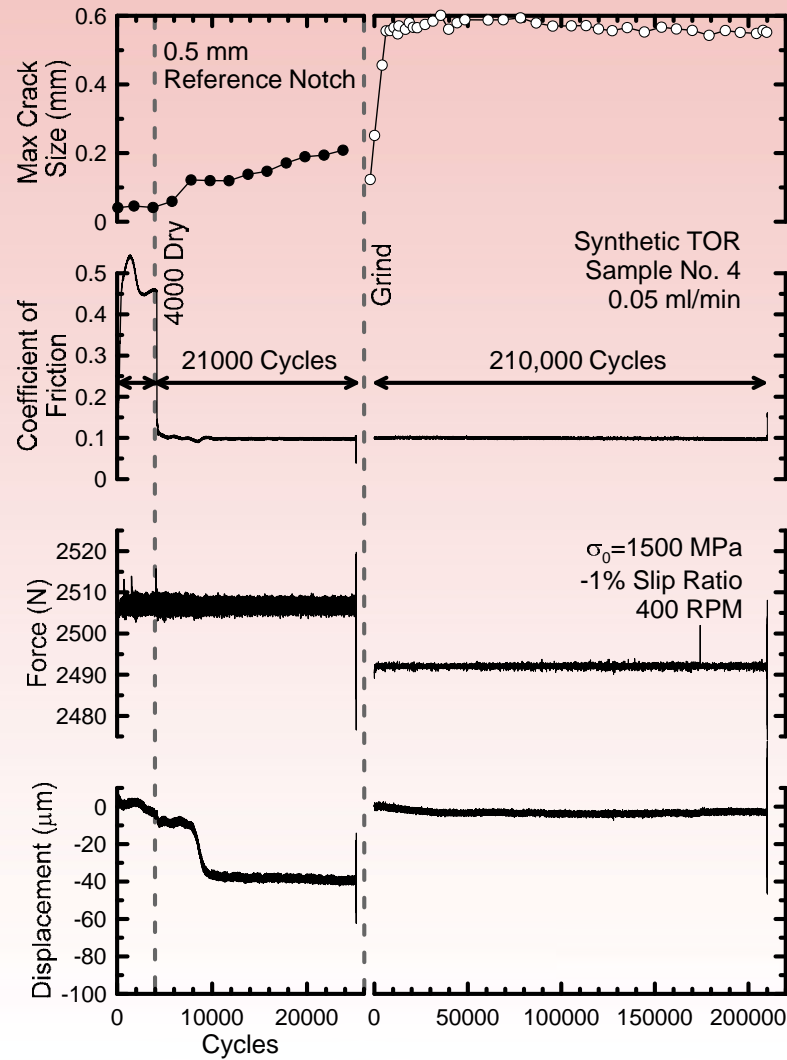
Water Based TOR Part 2



Synthetic TOR Part 2



Synthetic TOR Part 2



210K Lubricated

Water Based TOR



Synthetic TOR

